

THE IRON AND COAL DISTRICTS OF ALABAMA. No. III.

Leaving now the district in which ironmaking is being so rapidly developed, and turning again to the coalfields as affording fuel for export to the neighbouring States or abroad, the city of Tuscaloosa—an old Indian name signifying Black Warrior—claims the principal attention. Situated on the Warrior river, in the county of Tuscaloosa, fifty-five miles south-west of Birmingham, on the Western side of the State of Alabama, this city was formerly the capital of the State. The construction of the Alabama Great Southern Railroad has been the means of opening up the commerce and productions of the district to the world. This road connects at Chattanooga with all the eastern and western lines, runs south-west *via* Tuscaloosa to Meridian, on the Mobile and Ohio Railroad, a distance of 300 miles, splitting open, as it were, the greatest mineral region in the world, and connecting at Meridian with the lines running *via* Vicksburg to New Orleans, as well as to Cairo, Illinois, thus securing connection by rail with all the principal cities of the continent. As from its situation and other natural advantages, Tuscaloosa is likely to take a leading place in the coal and iron industries of the United States and in the numerous manufactures that depend on these minerals for development, some description of the place and its resources may be interesting to the readers of THE ENGINEER. Birmingham has been already described, and though there are many points of similarity in the two places, Tuscaloosa has peculiar advantages of its own. Referring to the map given in THE ENGINEER of 5th June, it will be seen that Tuscaloosa is on the extreme southern edge of a coal basin of vast extent, and at the head of the steamboat navigation to the Gulf of Mexico. This coincidence is of immense importance to the town and district, for cheap water carriage gives it a supremacy over places depending only on railways, and in the great coming trade of the South—the export of coal—Tuscaloosa is likely to hold the leading place. About seventy miles below the city, the Warrior river is increased in volume by the Tombigbee, which joins it at Demopolis, and then, though still winding in its course, runs almost due south, till it is joined by the Alabama river about forty miles above Mobile, and thus increased discharges into the Gulf of Mexico at Mobile Bay—a magnificent harbour for the largest ships. The Warrior is at present navigable at all seasons only as high up as Demopolis, and only for about half the year can steamboats reach Tuscaloosa. But after careful surveys by Government engineers it is found that the obstacles to navigation can be removed at moderate cost, so as to give at extreme low water a minimum channel 80ft. wide and 4ft. deep, amply sufficient for the transport of coal in barges towed by steamers. An example of this system is seen at Pittsburg, where coal is sent by the river Ohio and Mississippi in fleets of barges to New Orleans. Pittsburg flourishes with this trade, although the navigation is closed by ice for three or four months in every year, while the Warrior river is never so stopped. Following the plan that has prevailed in the United States for many years, national funds are appropriated to the improvements of navigable rivers, and just as the locks and canals were so constructed to open the Ohio river to navigation past the falls and rapids at Louisville, Kentucky; so will the necessary works be soon carried out on the Warrior river. The people have the more confidence in this since the election of Mr. Cleveland as President, and the return to power of the Democratic party, who sympathise more keenly with the development of the Southern States than did the late Government.

North of Tuscaloosa the river penetrates for fifty miles into the very heart of the coal country, and if the impediments to the southern navigation be removed as just mentioned, then the upper river will almost certainly be improved by private enterprise, and the coal measures of Alabama be more opened out than can be effected by all the railways existing or under construction in the State. The coal measures of the Warrior basin embrace an area of over 3000 miles; the basin, exceeding sixty miles in width, is traversed by the Warrior river almost centrally from north-east to south-west through its most productive part. The strata as they approach the river become quite horizontal, affording exposures of bituminous coal frequently from 10ft. to 20ft. thick in the beds of the tributary creeks, and often in the bed of the river itself, so that the getting of the coal and the placing it on barges for transport can be effected at low cost. In dealing with so vast an area of coal as is here indicated figures become bewildering, and it is idle to speculate on a future in which, according to moderate estimates, two million tons of coal per annum can be obtained practically in perpetuity. But it is significant to note in the report of the geologists who have explored this region that all this may be obtained from the seams of coal already penetrated, the lowest of which is not, perhaps, more than 100ft. below the surface. But underlying the lowest of the seams yet discovered there is a thickness of several hundred feet of coal measures wholly unexplored, which, when penetrated, will no doubt add several more to the number. Remembering in regard to this that as the earth is pierced deeper the coal seams, as a rule, become thicker, harder, and better, the mind in contemplating these several hundred feet of underlying coal measures is led to a conception of probabilities—not to say of certainties—that are wholly beyond the reach of computation.

The cost of transportation by barges to Mobile is estimated by Ohio river experts at 75c. (3s.) per ton, making the total cost of coal at Mobile 2 dols. 25c. (9s. 4d.) or, with a safe margin for contingencies, say 3 dols. (12s. 6d.) per ton. An inspection of these rates shows that the Warrior coal can be delivered on the Gulf at prices that defy competition from any quarter. Hence it is destined to supply the entire demand of the Gulf ports, and of the steam marine trading between them; and once embarked at low cost, Alabama coal is likely to supersede English coal now shipped in large quantities to the West Indies and South America.

Tuscaloosa is a bright, cheerful town, with more to

alleviate a prosaic trading life than is usual in American cities. In 1831, only twelve years after Alabama was admitted into the Union, Congress granted an ample endowment for the use of a seminary of learning. The University of Alabama was located by the General Assembly in the suburbs of the city. For more than fifty years, therefore, this place has enjoyed the inestimable advantages of having at its door a college of the highest grade, with a full staff of learned professors. Hundreds of young men have been students and graduates of this university, among whom are some of the ablest and most distinguished men of the Southern States. The university buildings were burnt by Sherman's army at the end of the Civil War, not even the books from the library being allowed to escape. Congress has since granted endowments from the public lands, and under new and favourable auspices the university has been in successful operation for some years, and this summer will witness the completion of additional blocks of buildings, increasing greatly the accommodation for professors and students.

The routine of the schools is of a *quasi* military kind; all the students wear soldiers' uniform; they are armed with rifles, have regular regimental drill, and do sentry duty by turn. Technical instruction seems to have a leading place, and the students are fortunate in having among their teachers the State Geologist, Mr. Eugene A. Smith, whose reports on the geology and mineralogy of Alabama have become so widely known, and who occupies his summer holidays by making new explorations. One of the benefits obtained by the recent enlargements of the college is the greater space accorded to laboratory teaching. About a mile from the university is the State lunatic asylum, pleasantly situated in park-like grounds, and conducted after the modern humane fashion usual in England. As typical of the mining facilities of the district, it may be mentioned that coal is dug out on the premises by the inmates and illuminating gas manufactured for use in the asylum and in the neighbouring university. Just at the end of the main street of the town the river Warrior is crossed by an iron bridge constructed of trussed girders of the type so common in America, namely, deep, light, and, to English eyes, weak; but if the traffic demands it this will soon be replaced or added to by a modern and stronger structure. On the opposite bank of the river is the only coal pit as yet opened close to the town, but the ease and cheapness with which the coal is obtained and put into barges exemplifies the facilities for future trade.

Following on a small scale the example set so largely in Georgia and Carolina, where numerous cotton factories have been established, Tuscaloosa at present boasts only of one cotton mill, which already forms the nucleus of the manufacturing suburb of Cottondale. Here, with the raw material grown close at hand, with a potential water power available, though at present unimportant, because of the cheapness of coal, and with railway and river transport to customers north and south, the future prospects of the trade are most promising. There is also an oil mill, and the comparatively new industry of crushing cotton seed has every opportunity to flourish in the future. The timber resources of Alabama seem almost endless, for the forest lands south of the city may be measured literally by hundreds of miles. For many miles the primeval forests of yellow pitch pine, poplar, ash, walnut, oak, cedar, cypress, and other valuable timbers are unexcelled in the country. Some of the pine lands will yield from 50,000ft. to 100,000ft. of timber to the acre, and for timber alone parts of these forests are worth from 50 to 100 dols. per acre, but when it is taken into consideration that much of this property is underlaid with seams of coal varying from 3ft. to 5ft. in thickness, the real ultimate value of the land may be realised, though it may now be bought at almost nominal prices.

Thus supplied by bountiful nature with coal and timber, with iron from the neighbouring city of Birmingham, and with the transport facilities already described, Tuscaloosa, with its genial climate, affords all that is required for mechanical trades of all kinds. Car building for the Southern railroads, agricultural implements of all kinds, and the numerous minor industries requiring wood and iron, would all thrive if commenced by those who understand these trades. But the cloud that has enveloped the South since the war is only now lifting; not only capital is wanting, but mechanics of all kinds, and fair Tuscaloosa, with her wide tree-lined avenues, must be patient yet awhile, till her countrymen from the North and the emigrants from beyond the sea recognise the opportunities she affords. And although some of the principal industries would probably flourish best if started on a large scale, there is ample opening for young men of enterprise and small means. Foundries, machine shops, agricultural implement factories, and the numerous subsidiary trades that find employment in a manufacturing district, would all succeed here, if managed with industry and perseverance, and no emigrants are better received than those from the old country.

Having now described the local advantages and peculiar features of the Alabama mineral districts, it may be convenient to quote the opinion of some of the well-known leaders of the iron trade in America and England. As long ago as 1871 Mr. Abram S. Hewitt, of New York, said of Northern Alabama, "it is, in fact, the only place upon the American Continent where it is profitable to make iron in competition with the cheap iron of England, measured not by the wages paid, but the number of days' labour which enter into its production." Then, after stating the cost of making iron at Cleveland in England, where the distance of the coal and the ore from the furnaces averages about twenty miles, Mr. Hewitt says:—"In Alabama the coal and the ore are in many places within half-a-mile of each other, and the cost of the iron is only about ten days' labour to the ton, or not far from the labour cost in Cleveland. Throwing aside, then, all questions of tariffs for protection, here is a possibility upon the American Continent of producing iron at as low a cost in labour as in the most favoured region of the world, and allowing for the expense of transportation to compete with them, paying a higher average rate of wages than is paid

in Great Britain." An equally eminent English authority, Mr. Lowthian Bell, who has made many visits to the mineral regions of America, said, in 1875, before the Iron and Steel Institute of Great Britain, that "the undeveloped resources of Alabama, Tennessee, and Georgia would prove a match for any part of the world in the production of cheap iron." The year 1875 is a long time ago in so rapidly growing a country as the United States, and it would be interesting to know if Mr. Bell has visited Alabama since that date, and before writing the following in his book, published last autumn, "The Principles of the Manufacture of Iron and Steel." Commencing on page 100 of this book, Mr. Bell says:—"It seems to me that so long as the Northern States are dependent on their present mining resources as regards ore, it is futile to hope for any export trade from that division of the Union. On the contrary, the ironmasters of the North must prepare themselves for importations, not from Europe, but from a quarter against which the present legislative constitution of the States will afford no protection. The quarter alluded to, of course, the Southern States. Very trifling extensions of the present railways will place the whole of Tennessee, Alabama, and Georgia in direct communication with the Tennessee river. I understand one impediment only exists which impedes free navigation. This removed, the Mississippi and Ohio will become accessible from those States by steam navigation. The distance from a central point, say Chattanooga, to Pittsburg by river is probably 1000 miles, for which the freight will not exceed that from Great Britain. In these Southern States coal can be worked nearly as cheaply as at Connellsville, while the labour on the whole of the ore entering into the manufacture of a ton of iron is not more than that expended on the extraction of a single ton of ore near Marquette. Besides this there is the fact that the bringing of the minerals together in the Northern States often costs 30s. to 40s. per ton of iron made. With these elements of cost, it seems impossible to deny that, in the absence of fresh ore discoveries in the North, time alone is required to produce a considerable change in the seats of the American iron trade. This state of things naturally suggests the inquiry as to the ability of the South to enter the markets of the world in competition with Great Britain. It cannot be disputed that up to this time pig iron has never been produced in Alabama, or in its vicinity, within some shillings per ton of the price at which it can be made from Cleveland ironstone in England. The removal of difficulties which always beset the introduction of new industries may partly equalise these differences; but by that time labour probably will no longer be procurable in the Southern States upon so much lower terms than it commands in the North. Be this, however, as it may, there remains the insurmountable difficulty of the cost of transport to the chief iron-consuming populations in the world, viz., to those of Europe. The nearest point of the Alabama mineral field cannot be short of 150 miles from the sea-board. Admitting the carriage from the works to be done for $\frac{1}{2}$ d. per ton per mile, this added to the Atlantic freight would probably entail a cost of 20s. per ton of iron delivered on the shores of Great Britain or of northern Europe above that paid by ourselves or by Germany. This extra charge for freight no doubt would be reduced when competing with us for the custom of the Mediterranean ports or those of Asia, South Africa, Australia, &c. Accepting Cleveland as a standard of comparison, the expenses of manufacturing pig iron included about 7s. 9d. for railway charges and 3s. 9d. for royalty dues. These together amount to 11s. 6d., or nearly 33 per cent. of its entire cost; whereas in the Southern States the two items are not half the sum just named. Can the English railway companies abate their charges, and will the English landowners be satisfied with more moderate royalties? The relative position of Great Britain and the Southern States of America may be materially altered, not immediately, but within a few generations. The capability of the Durham coal-field to furnish cheap fuel to the ironworks will be gradually curtailed; and as this takes place coal and coke will have to be brought from greater distances. At present the States in question may be regarded as virgin ground, in which iron ore and coal, as I understand, exist in sufficient abundance to endure long after the north-eastern coal-fields of England are exhausted."

It will be seen from the foregoing remarks that Mr. Lowthian Bell does not refer to the possibility of water carriage south except by the somewhat circuitous route of the Tennessee River to the Ohio and the Mississippi. But if not from Birmingham, then from the district between it and Tuscaloosa, the navigation by the Warrior River to Mobile is likely soon to be opened. But even pending this development, the Georgia Pacific and other of the railways are encouraging the trade northwards by way of Savannah, and thence by sea to Philadelphia and New York. The sale of Alabama iron in these cities has been encouraged by the low rates of carriage of $\frac{1}{3}$ ths of a mile—equal to one farthing—per ton per mile. Looking at this question of carriage, it would seem probable that the Tennessee River will be the natural outlet for Sheffield and all places between that town and Birmingham, and that the Warrior River will serve for the coal export from Tuscaloosa. Birmingham itself will have both these routes available, but must always depend mainly on the numerous railways that radiate in all directions from her furnaces and mills. Mr. L. Bell shows that great as are the natural resources and advantages of Alabama, competition with English iron and coal in England is not to be feared until our coal-fields approach exhaustion. But it is the export trade of Great Britain that is threatened, for if only coal and iron can be put on board ship cheaply at New Orleans, Mobile, Savannah, and Charleston, then water carriage to all parts of the world will be as cheap, according to distance, as from England.

It will be noticed in the preceding articles that no mention has been made of steel. It is not by giving the name of Sheffield to an ironmaking town that steel is to be produced there, and at present no suitable ore has been found in Alabama. It is true that occasional analyses show ore sufficiently free from phosphorus and otherwise

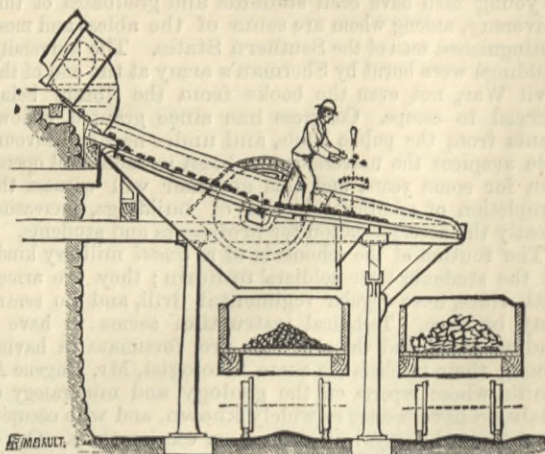
good, but no continuous supply has yet been obtained. If, however, good coke can be produced, and if a market grows up southward for the finished product, then there is no reason why steel-making ore should not be brought to the district, as is done to Pennsylvania. Perhaps the most likely source would be from Missouri, which State already supplies the steel works at St. Louis and elsewhere as a return cargo in the steamers that have taken iron and coal north from Sheffield. But in the district itself there are sanguine expectations that non-phosphoric ore will yet be found nearer at hand, and in sufficient quantity. There are enormous deposits of rich iron ore in Georgia waiting exploitation, and some of it is said to show only traces of phosphorus. In West Virginia it is positively stated that steel-making ore exists in large quantities, but as the explorers wanted the mineral to supply Pennsylvania furnaces, the distance was too great. In Virginia itself, or south of it, will be the proper place for using the ore. It must be remembered that there are already established in the United States, steel rail mills with a total annual capacity of more than one and a-half million of tons, and that this quantity can only be consumed by the continual making of new railways, so that any Southern steel works will have to compete on rigorous terms with the works already in operation, although for railways in the Southern States and in Mexico they would have a geographical advantage over those now established in the North.

MISCELLANEOUS MACHINERY AT THE INVENTIONS EXHIBITION.

AMONGST the more important of the machinery illustrated by drawing and by model is the coal-loading machinery by Mr. James Rigg, of Queen Victoria-street. Competition in the coal as in most other trades has of late years directed the attention of those chiefly interested, merchants as well as coalowners, to the most economical and expeditious method of conducting them. Of the many improvements introduced for this purpose, machinery for tipping with the least damage to the coal stands first, and in illustration of what has been done in this direction we describe Mr. Rigg's system. The value of coal is in almost all cases dependent upon the size of the pieces as received by the consumer and from its arrival in the truck or tram at the pit's mouth; each occasion of its being transferred from one means of conveyance to another has the effect of more or less injuring it. Though the more friable house coals particularly sustain damage by the abrasion—inseparable from transit in railway trucks—the more important occasions are in the tipping and screening at the pit and in the loading at the shipping port. At the pit many methods have from time to time been devised more or less with a view to effecting the above objects, necessarily varying with the character of the coal in the different districts of the country. Of the various means of tipping the pit tub,

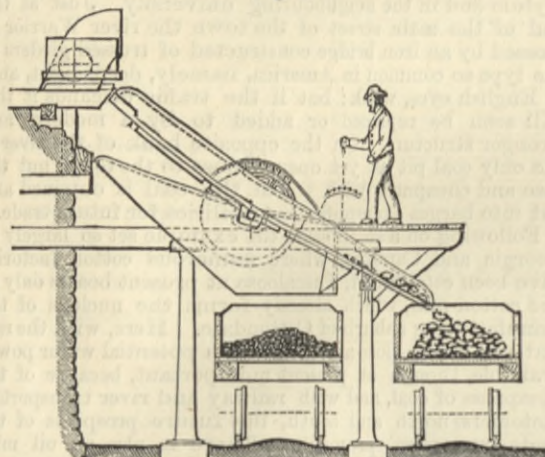
on the coal arriving at the top of the screen. Fig. 1 gives a perspective view of these tipping machines working in conjunction with the curved balanced screen, which are specially adapted to the cleansing of slag or other impurities from the best house coal, these usually being of a very friable description. The diagram, Fig. 2, shows this screen in its normal position receiving a tub

Fig. 2



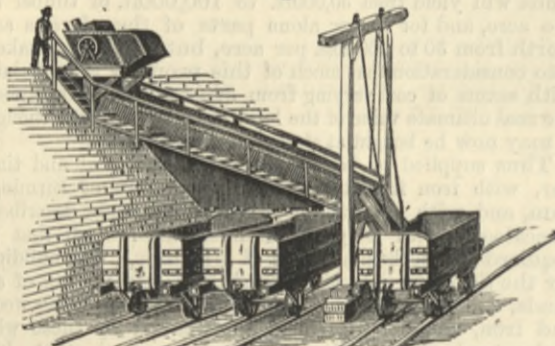
load of coal, which, passing down the more inclined part of it, is gradually brought to rest upon the lower screen bars, and it has here so distributed itself as to permit of its ready examination, and the removal of all objectionable matter, for which there is accommodation as shown in the platform in Fig. 1. The round coal being thus ready for

Fig. 3



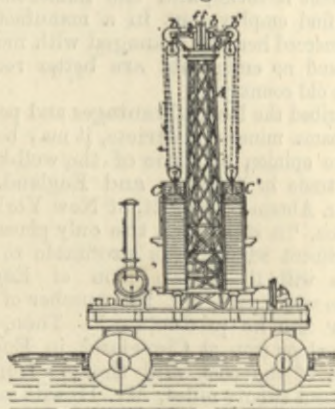
loading into its truck, the preponderance due to its weight causes the screen, on the brake being released, to be depressed to an angle found just sufficient for the purpose, as shown in Fig. 3. The brake is again released, and the screen returns to its normal position to receive another load, these operations having been performed during the

Fig. 4



period necessary for changing the tubs in the tip above. The slack is received in the fixed hopper shown, and thus passes into its own truck. One special feature in regard to these screens consists in the manner in which the steel screen rails are divided into several lengths and ranges, and so pitched that the spaces between one set of bars are in a direct line with the bars of the range preceding them, thus greatly facilitating the passage of the slack into its

Fig. 10



hopper. A short screen on this principle produces more efficient work than a long one, having continuous bars on the usual system. Fig. 4 gives a perspective view of a fixed double screen for making round coal nuts, and slack from coal which, though requiring to be dealt with carefully, does not contain the impurities, for the removal of which the former screen is constructed. The steel rails in those are usually larger and continuous. Balanced doors are provided under the hoppers, and a large one to convey

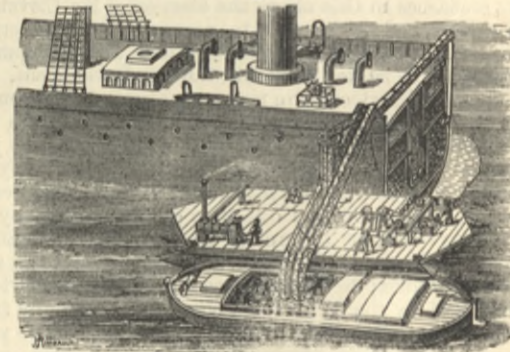
the round coal into its truck. A fixed screen is also represented in Fig. 5, this sketch being more especially intended to show the manner in which the coal is retained in the tipping machine, which, had it been in an ordinary open tippler, would before it attained this position have fallen down the screen and been subject to consequent injury and deterioration in value. As Mr. Rigg has erected over four hundred of his coal-tipping machines for various coal districts in this country and abroad, he has necessarily arranged a large variety of screens to suit the special requirements of the coal to be dealt with. Those referred to above with the single fixed screen for making round coal and "through" slack may, however, be taken to represent their leading characteristics. The system of loading coal under control of a brake, and by utilising

Fig. 5



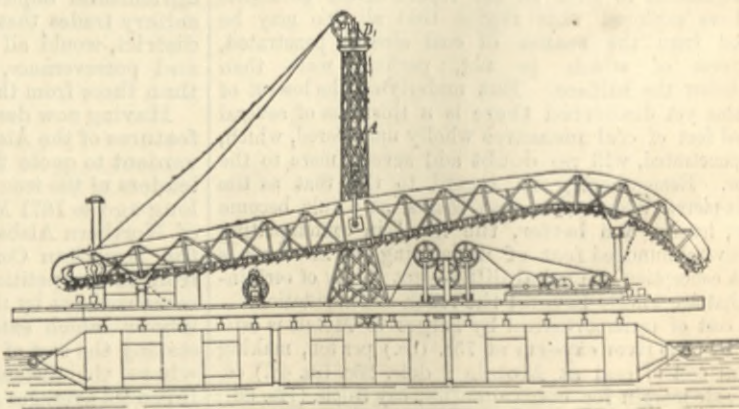
gravity when available, in preference to hydraulic or other power, has also been applied with success to the tipping of end door, narrow gauge—4ft. 8½in.—railway trucks, and one of these machines recently supplied for the West Bank Dock, near Widnes, is working with good results. The views given of this machine are Fig. 6, end elevation, Fig. 7, side elevation, and Fig. 8, plan. The oscillating platform of this machine is strongly constructed of wrought iron, and is so balanced that all trucks of medium length when placed in the machine tip it forward and cause its action, under control of its brake, the projecting spout or shoot parallel to the floor of the wagon conveying the coal into the hatch, thus not only avoiding the intervention of a separate shoot, but materially reducing the breakage which occurs in the passage of coal from a truck when otherwise tipped above the ordinary adjustable shoots. Though a fair average may be determined as the length of a coal truck, and this tip so balanced as to suit it, it is obvious that in the few cases in which the trucks may be

Fig. 9



found extremely short or correspondingly long, gravity will act in one direction only, and to complete the operation of tipping in such cases, the winch shown in the two elevations may be used. The machine illustrated is used principally for loading small coasting vessels, and to give increased facility for changing hatchways, the projecting shoot is balanced and arranged to rise as shown in Fig. 7. The relative positions of shoot and vessel are seen in the

Fig. 11



separate outline diagram, Figs. 6, 7, and 8, on next page. The capacity of these machines is necessarily limited by the pace at which the cargo loaded can be trimmed in the hold, and were this not so, its regular performance of 1000 tons per day of ten hours could easily be increased. Mr. Rigg's floating elevator, Figs. 9, 10, and 11, is intended to increase the pace at which coal is loaded in this country and abroad from barges both as "cargo" and for "bunker coal." Its construction and operation will be readily

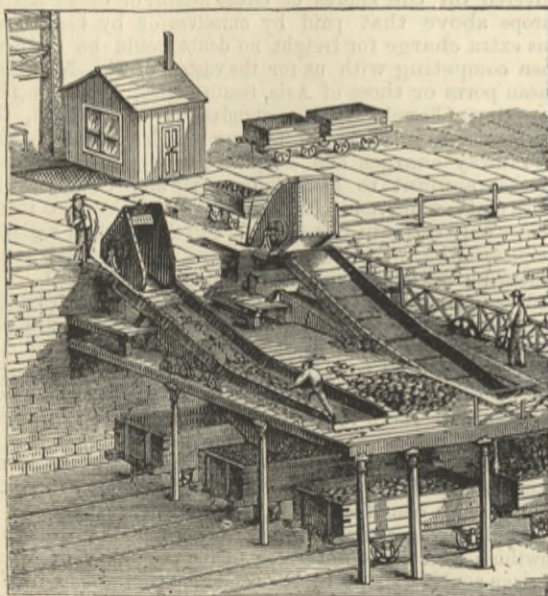


Fig. 1

perhaps the best known are the tram of South Wales and Monmouthshire, having a simple bar or door hinged at their ends, the tram assuming an angle of about 35 deg. by its fore wheels falling into recesses at the head of the screen or by the whole tram being placed upon an oscillating platform so arranged as to allow of its assuming such an angle as usually permits of the coal clearing itself. The so-called "box tubs"—that is, tubs having no doors—are also variously tipped in balanced and unbalanced frames backwards, forwards, and sideways on to the screen, sometimes under control of a brake. In cases also where the coal is not loaded above the top of the tub, a horizontal door is sometimes shut down upon it and not released until the tub is inverted over the screen. All these methods, however, cause unnecessary breakage, and all deliver the coal in a mass at the top of the screen, this portion of the screen being consequently almost useless for its intended purpose of separation.

The tipping machine which we first notice is represented in the first five figures in the commencing, intermediate, and concluding positions of delivering coal upon a screen under the easy control of a man or boy, the rotating bonnett or box being so balanced that the position of the centre of gravity dependent upon the tub being loaded or discharged causes it, under control of the brake, either to tip forward or return. Within this bonnett is a horizontal hinged door, which, notwithstanding any pace at which the machine may be allowed to work, though checking the tendency of the coal under such circumstances from leaving too fast, at the same time yields to it, and thus combines with the bonnett or shoot in spreading the coal and causing the process of separation to commence at once

understood by reference to Fig. 9, which gives a perspective view of one of these elevations bunking coal from a lighter into a large steamer. It will be seen on reference to Fig. 9, as well as 10 and 11, that the pontoons, which

athwart in Fig. 9. The power for this purpose is obtained from the pair of winding engines on the deck, which also drive the buckets by means of an endless steel wire rope. The suspended girders are hauled into posi-

illustrated in Fig. 17, the pace at which they travel being under control of a brake, and motion being imparted by gravity, are raised either by the ship's tackle or its own winch, should other power not be available. As the

Fig. 6

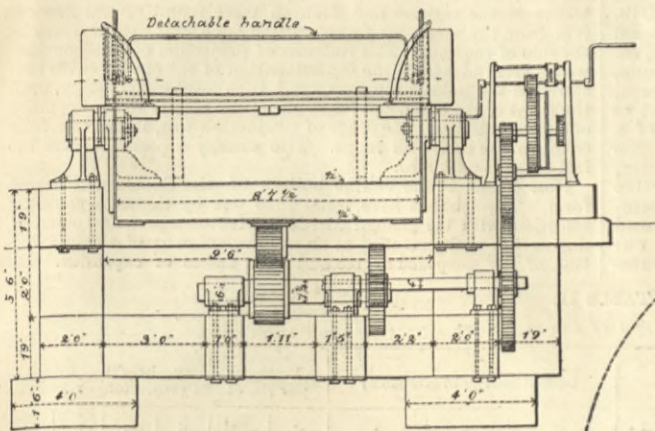


Fig. 7

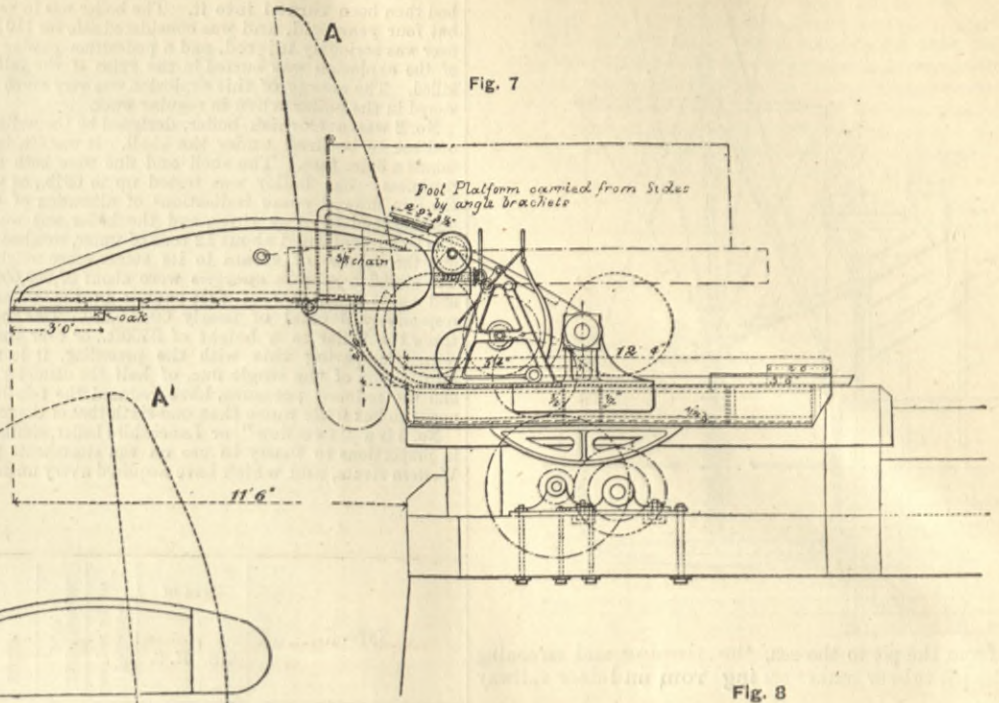
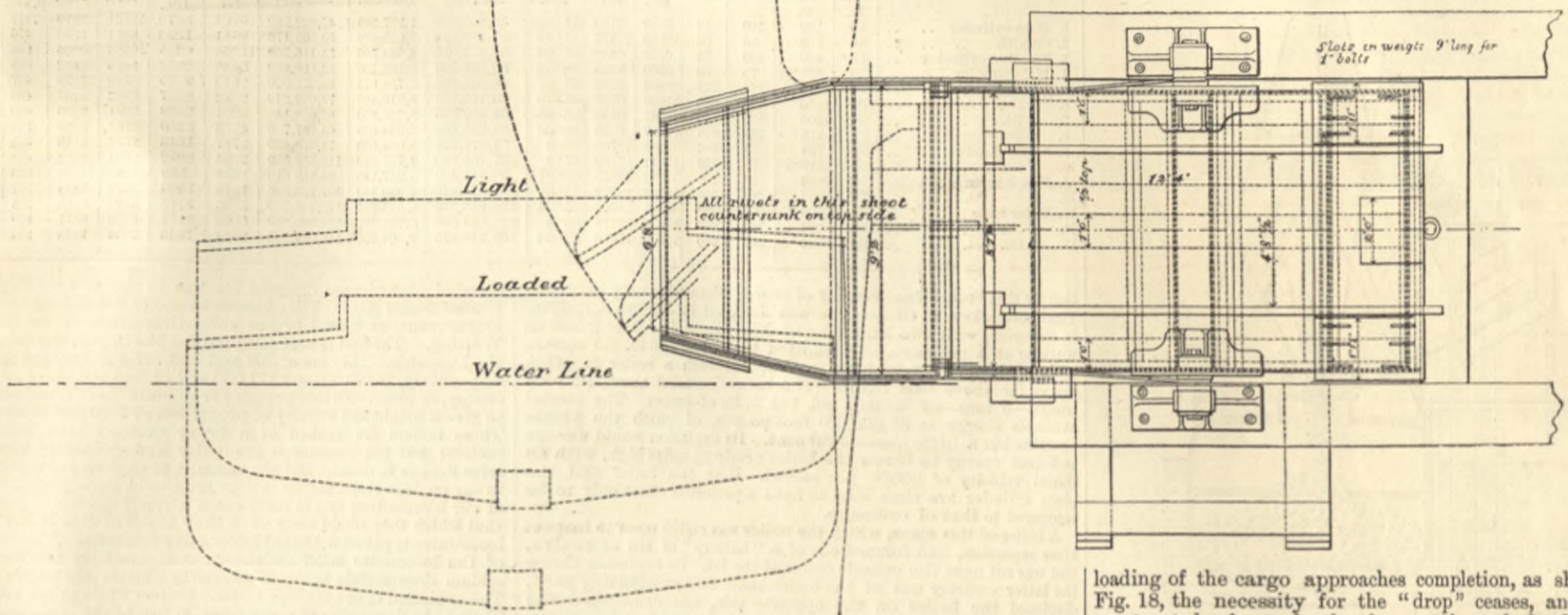


Fig. 8



carry a deck of about 24ft. in beam, support about the centre of this deck a lattice tower, and upon this is a revolving cast iron head, from which is suspended a pair

tion to suit the lighter and steamer by means of the hand winches shown in Fig. 9, and some idea may be formed of the facility with which they are raised up and placed

loading of the cargo approaches completion, as shown in Fig. 18, the necessity for the "drop" ceases, and, being constructed of steel, the framework and buckets are

Fig. 12

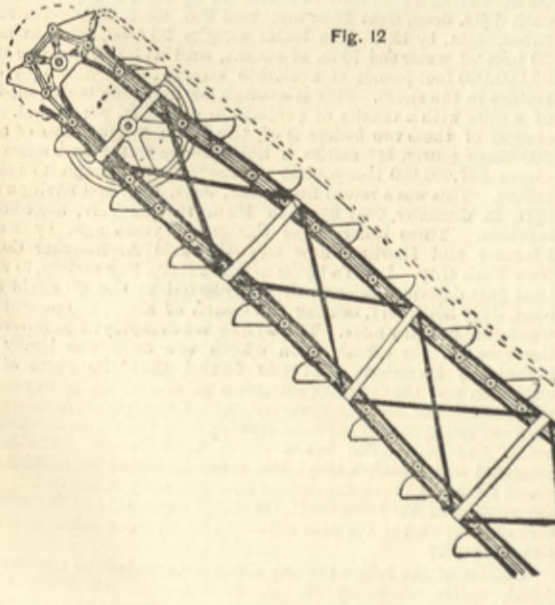


Fig. 13

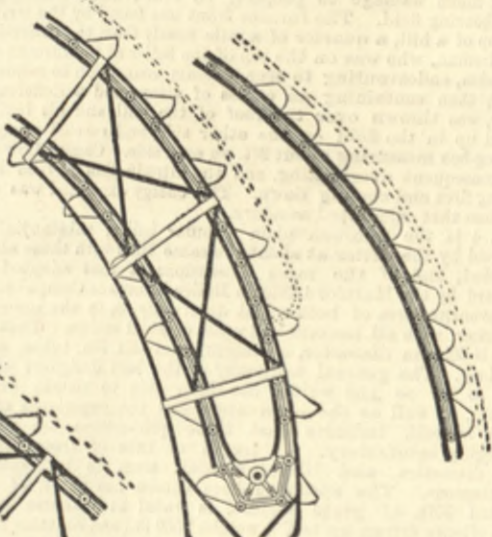
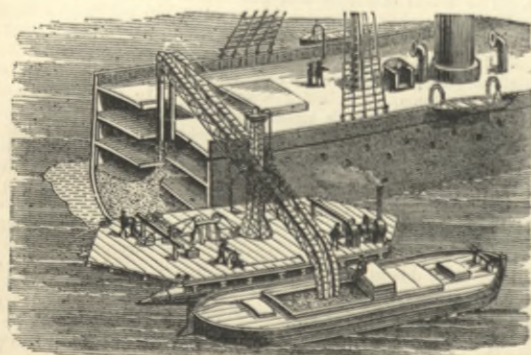


Fig. 14

of steel lattice girders each containing an endless chain of buckets or trays, as shown in Figs. 12 and

Fig. 15

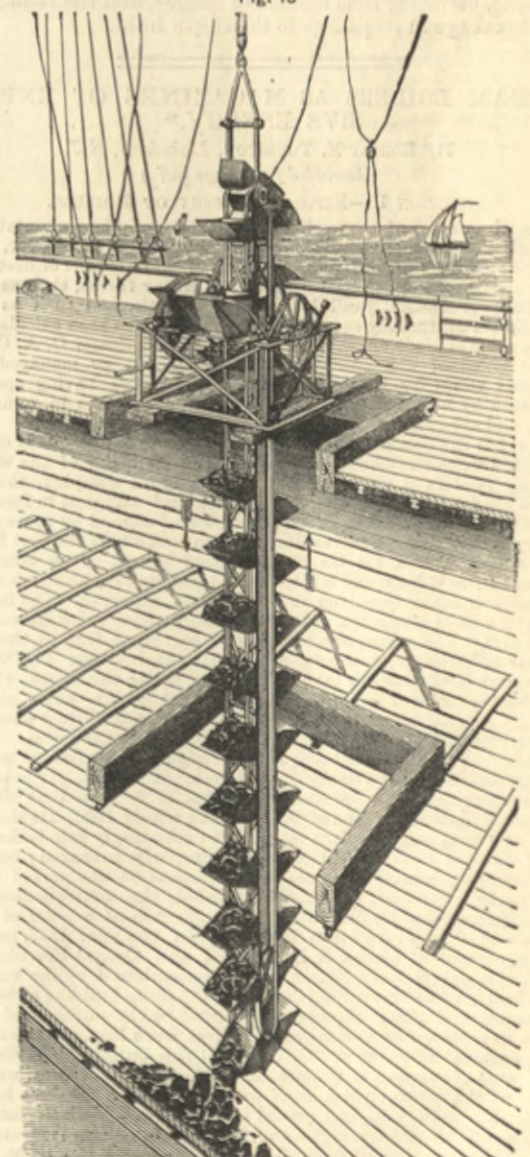


13. These girders are raised from the fore and aft positions, shown in Figs. 10 and 11, to the direction

athwart when it is stated that the former operation has frequently been performed in seventy-five and the latter ninety seconds. An illustration, Fig. 15, represents one of these elevators as used for loading grain, and to which the buckets, Fig. 14, were applied. A pair of 7in. by 14in. winding engines have thus raised grain at a pace exceeding 150 tons per hour. These buckets fill by gravity, which is not the case with coal, the quantity being limited to the filling capacity of, say, eight men in close proximity to the foot of the elevating coal trays or buckets, or say about 60 tons per hour. It may, perhaps, be hardly necessary to add that the cost of the work under this system, as compared with manual labour, is very small.

The automatic drop illustrated in Figs. 16, 17, and 18 is intended to reduce the breakage of coal in its passage, as is now usual, without check from the vessel's hatch to the bottom of the hold. The greatest injury takes place at the commencement of loading the cargo, and the continuous chain of buckets shown in Fig. 16 conveys it to the bottom of the vessel's hold, thus avoiding this fall. The apparatus consists of this chain of buckets, which intercept the coal as received from the ordinary shoots

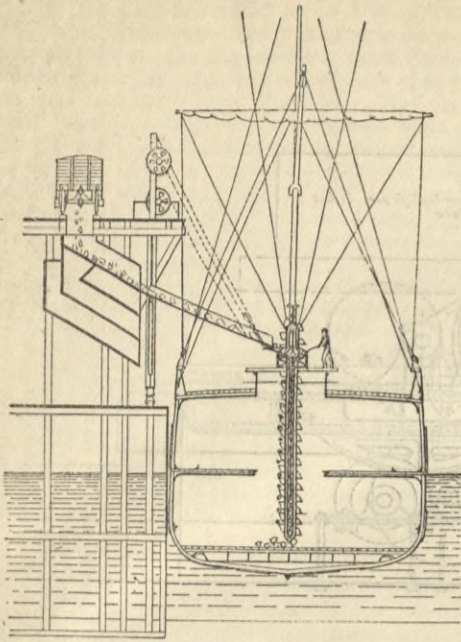
Fig. 16



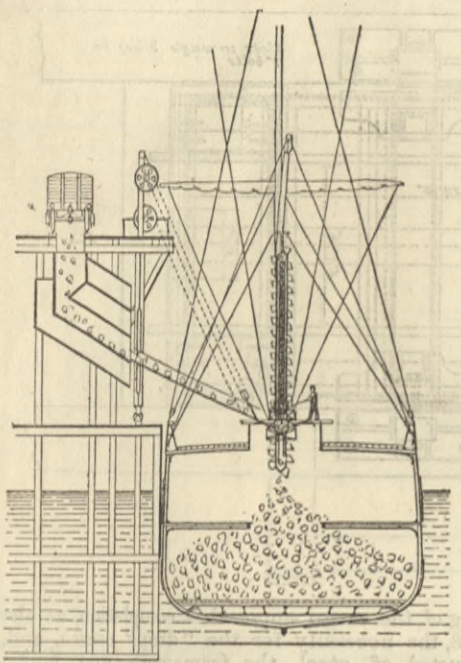
readily removed from this position. Their total weight is about 40 cwt.

The machinery we have illustrated thus makes up a complete set for accomplishing the various steps in the removal

Fig. 7



of coal from the pit to the sea, the tinning and screening from the pit tubs or trams facing from end-door railway



trucks, elevating from lighters or barges, and the reduction of breakage in its passage to the ship's hold.

STEAM BOILERS AS MAGAZINES OF EXPLOSIVE ENERGY.*

By ROBERT H. THURSTON, Hoboken, N.J. (Concluded from page 507.)

SECTION II.—EXPLOSIVE ENERGY OF BOILERS.

In illustration of the results of application of the computations which have been given in the preceding section of this paper, and for the purpose of obtaining some idea of the amount of destructive energy stored in steam boilers of familiar forms, such as the engineer is constantly called upon to deal with, and such as the public are continually endangered by, Table II. has been calculated. This table is made up, with the assistance of Professor C. A. Carr, from notes of dimensions of boilers designed, or managed, at various times by the writer, or in other ways having special interest to him. They include nearly all the forms in common use, and are representative of familiar and ordinary practice.

No. 1 is the common, simple, plain cylindrical boiler. It is often adopted when the cheapness of fuel or the impurity of the water supply renders it unadvisable to use the more complex, though more efficient, kinds. It is the cheapest and simplest in form of all the boilers. The boiler here taken was designed by the writer many years ago for a mill so situated as to make this the best form for adoption, and for the reasons above given. It is 30in. in diameter, 30ft. long, and is rated at 10-horse power, although such a boiler is often forced up to double that capacity. The boiler weighs a little over a ton, and contains more than twice its weight of water. The water, at a temperature corresponding to that of steam at 100 lb. pressure per square inch, contains over 46,600,000 foot-pounds of available explosive energy; while the steam, which has but one-fifth of 1 per cent. of the weight of the water, stores about 1,300,000 foot-pounds, giving a total of 48,000,000 foot-pounds, nearly, or sufficient to raise 1 lb. nearly 10,000,000 miles. This is sufficient to throw the boiler 19,000ft. high, or nearly four miles, and with an initial velocity of projection of 1111ft. per second. Comparing this with the succeeding cases, it is seen that this is the most destructive form of boiler on the whole list. Its simplicity and its strength of form make it an exceedingly safe boiler, so long as it is kept in good order and properly managed; but if, through phenomenal ignorance or recklessness on the part of proprietor or attendant, the boiler is exploded, the consequences are usually exceptionally disastrous. The explosion of a boiler of this form and of the proportions given here, in the year 1843, in the establishment of Messrs. R. L. Thurston and Co., at Providence, R.I., through mismanagement, is well remembered by the writer. The boiler house was entirely destroyed, the main building seriously damaged, and a large expense was incurred in the purchase of new tools to replace those destroyed. No lives were lost, as the explosion occurred after the workmen had left the building. A similar explosion of a boiler of this size occurred some years later, within sight of the writer, which drove one end of the exploding boiler through a 10in. wall, and several hundred feet through the air, cutting off an elm tree high above the ground where it measured 9in. in diameter, partly destroying a house in its further flight, and fell in the street beyond, where it was found red hot immediately

after striking the earth. Long after the writer reached the spot, although a heavy rain was falling, it was too hot to be touched, and was finally, nearly two hours later, cooled off by a stream of water from a hose, in order that it might be moved and inspected. It had been overheated, in consequence of low water, and cold feed had then been turned into it. The boiler was in very good order, but four years old, and was considered safe for 110 lb. The engineer was seriously injured, and a pedestrian passing at the instant of the explosion was buried in the ruins of the falling walls and killed. The energy of this explosion was very much less than that stored in the boiler when in regular work.

No. 2 was a Cornish boiler, designed by the writer about 1860, and set to be fired under the shell. It was 6ft. by 36, and contained a 36in. flue. The shell and flue were both of iron 3/4 in. in thickness. The boiler was tested up to 60 lb., at which pressure the flue showed some indications of alteration of form. It was strengthened by stay rings, and the boiler was worked at 30 lb. The boiler contained about 12 tons of water, weighed itself 7 1/2 tons, and the volume of steam in its steam space weighed but 3 1/2 lb. The stored available energies were about 57,600,000 foot-pounds, and about 2,000,000 of foot-pounds in the water and steam, respectively, a total of nearly 60,000,000. This was sufficient to throw the boiler to a height of 3500ft., or over three-fifths of a mile. Comparing this with the preceding, it is seen that the introduction of the single flue, of half the diameter of the boiler, and the reduced pressure, have reduced the relative destructive power to but little more than one-sixth that of the preceding form.

No. 3 is a "two-flue" or Lancashire boiler, similar in form and in proportions to many in use on the steamboats plying on our Western rivers, and which have acquired a very unenviable reputa-

usual way.* On the occasion of such an explosion which the writer was called upon to investigate, in the course of his professional practice, the engine was hauling a train of coal cars weighing about 1000 tons. The steam had been shut off from the cylinders a few minutes before, as the train passed over the crest of an incline and started down the hill, and the throttle again opened a few moments before the explosion. The explosion killed the engineer, the fireman, and a brakeman, tore the fire-box to pieces, threw the engine from the track, turning it completely around, broke up the running parts of the machinery, and made very complete destruction of the whole engine. There was no indication that the writer could detect of low water; and he attributed the accident to weakening of the fire-box sheets at the lower parts of the water-legs by corrosion. The use of water-grates, the insertion of which produced some loss of strength at the fire-box, may have had something to do with it, however. The bodies of the engineer and fireman were found several hundred feet from the wreck, the former among the branches of a tree by the side of the track. This violence of projection of smaller masses would seem to indicate the concentration of the energy of the heat stored in the boiler, when converted into mechanical energy, upon the front of the boiler, and its application largely to the impulsion of adjacent bodies. The range of projection was, in one case, fully equal to the calculated range. The energy expended is here the full amount calculated.

Nos. 9 and 10 are marine boilers of the Scotch or "drum" form. These boilers have come into use by the usual process of selection, with the gradual increase of steam pressures occurring during the past generation as an accompaniment of the introduction of the compound engine and high ratios of expansion. The

TABLE II. Stored Energy of Steam Boilers.

Table with 15 rows and 15 columns. Columns include Type, Area of (G. S., H. S.), Pressure, Rated power, Weight of (Boiler, Water, Steam), Stored energy in (available) (Water, Steam, Total), Energy per lb. of (Boiler, Total weight), Max. height of projection (Boiler, Total), and Initial velocity (Boiler, Total).

tion by their occasional display of energy when carelessly handled. That here taken in illustration was designed by the writer, 42in. in diameter, with two 1 1/4 in. flues of 3/4 in. iron, and is here taken as working at a pressure, as permitted by law, of 150 lb. per square inch. It is rated at 35-horse power, but such a boiler is often driven far above this figure. The boiler contains about its own weight—3 tons—of water, and but 37 lb. of steam. The stored available energy is 85,000,000 foot-pounds, of which the steam contains but a little above 5 per cent. Its explosion would uncase sufficient energy to throw the boiler nearly 2 1/2 miles high, with an initial velocity of 900ft. per second. Both this boiler and the plain cylinder are thus seen to have a projectile effect only to be compared to that of ordnance.

A boiler of this class, which the writer was called upon to inspect after explosion, had formed one of a "battery" of ten or twelve, and was set next the outside boiler of the lot. Its explosion threw the latter entirely out of the boiler-house into an adjoining yard, displaced the boiler on the opposite side, and demolished the boiler-house completely. The exploding boiler was torn into many pieces. The shell was torn into a helical ribbon, which was unwound from end to end. The furnace end of the boiler flew across the space in front of its house, tore down the side of a "kier-house," and demolished the kier, nearly killing the kier-house attendant, who was standing between two kiers. The opposite end of the boiler was thrown through the air, describing a trajectory having an altitude of 50ft., and a range of several hundred, doing much damage to property en route, finally landing in a neighbouring field. The furnace front was found by the writer on the top of a hill, a quarter of a mile nearly from the boiler-house. The fireman, who was on the top of the boiler at the instant of the explosion, endeavouring to open a steam connection to relieve the boiler, then containing an excess of steam and a deficiency of water, was thrown over the roof of the mill, and his body was picked up in the field on the other side, and carried away in a packing-box measuring about 2ft. on each side. Cause: Low water and consequent overheating, and the introduction of feed before hauling fires and cooling down. The energy expended was much less than that calculated as above.

No. 4 is the common plain tubular boiler, substantially as designed by the writer at about the same time with those already described, and of the same dimensions as that adopted as a standard by the Hartford Steam Boiler Insurance Company.* It is a favourite form of boiler, and deservedly so, in the opinion of the writer, with all makers and users of shell boilers. That here taken is 60in. in diameter, containing sixty-six 3in. tubes, and is 15ft. long. The general testimony of the best designers of this type, so far as the writer has been able to obtain definite opinions, as well as the observation and the experience of the writer himself, indicate that these proportions are usually thoroughly satisfactory. A length of tube of from fifty to sixty diameters, and liberal spacing, seem to be especially advantageous. The specimen here chosen has 850ft. of heating and 30ft. of grate surface, is rated at 60-horse power, but is oftener driven up to 75, weighs 9500 lb., and contains nearly its own weight of water, but only 21 lb. of steam, when under a pressure of 75 lb. per square inch, which is below its safe allowance. It stores 52,000,000 foot-pounds of energy, of which but 4 per cent. is in the steam, and this is enough to drive the boiler just about one mile into the air, with an initial velocity of nearly 600ft. per second. The common upright tubular boiler may be classed with No. 4.

Nos. 5-8 are two of the Baldwin and two of the Cooke locomotive boilers, of which drawings and weights are furnished by the builders. They are of different sizes, and both freight and passenger engines. The powers are probably rated low. They range from 15 to 50 square feet in area of grate, and from 875 to 1350 square feet of heating surface. In weight the range is much less, running from 2 1/2 to a little above 3 tons of water, and from 20 lb. to 30 lb. of steam, assuming all to carry 125 lb. pressure. The boilers are seen to weigh from 2 1/2 to 3 times as much as the water. These proportions differ considerably from those of the stationary boilers which have been already considered. The stored energy averages about 70,000,000 foot-pounds and the heights and velocities of projection not far from 3000ft. and 500ft.; although in one case they became nearly one mile and 550ft. respectively. The total energy is only exceeded among the stationary boilers by the two-flued boiler at 150 lb. pressure. The violence of the explosion of the locomotive is naturally most terrible, exceeding, as it does, that of ordnance fired with a charge of 150 lb. of powder of best quality, or perhaps 250 lb. of ordinary quality fired in the

selected examples are designed for use in the new vessels of the United States Navy. The dimensions are obtained from the Navy Department, as figured by the chief draughtsman, Mr. Geo. B. Whiting. The first is that designed for the Nipsic, the second for the Despatch. They are of 300 and 350-horse power, and contain, respectively, 74,000,000 and 112,000,000 of foot-pounds of available energy, or about 3000 foot-pounds per pound of boiler, and sufficient to give a height and velocity of projection of 3000 and above 400ft. These boilers are worked at a lower pressure than locomotive boilers; but the pressure is gradually and constantly increasing from decade to decade, and the amount of explosive energy carried in our modern steam vessels is thus seen to be already equal to that of our locomotives, and in some cases already considerably exceeds that which they would carry were they supplied with boilers of the locomotive type and worked at locomotive pressures. The explosion of the locomotive boiler endangers comparatively few lives and seldom does serious injury to property outside the engine itself. The explosion of one of these marine boilers while at sea would be likely to be destructive of many lives, if not of the vessel itself and all on board.

Nos. 11 and 12 are boilers of the older type such as are still to be seen in steamboats plying upon the Hudson and other of our rivers, and in New York harbour and bay. No. 11 is a return tubular boiler having a shell 10ft. in diameter by 23ft. long, two furnaces each 7 1/2 ft. deep, eight 15in. and two 9in. flues, eighty-five return tubes, 4 1/2 in. by 15ft. The boiler weighs 25 tons, contains nearly 20 tons of water and 70 lb. of steam, and at 30 lb. pressure stores 95,000,000 foot-pounds of available energy, of which 5 per cent. resides in the steam. This is enough to hoist the boiler one-third of a mile with a velocity of projection of 330ft. per second. The second of these two boilers is of the same weight, also of about 200-horse power, but carries a little more water and steam and stores 107,000,000 foot-pounds of energy, or enough to raise it 1900ft. This was a return flue boiler, 33ft. long and having a shell 8 1/2 ft. in diameter, flues 8 1/2 in. to 15in. in diameter, according to location. These boilers were designed, years ago, by Messrs. Fletcher and Harrison—now the W. and A. Fletcher Co.—of New York City. It was a boiler of the return flue variety, to which that just described belongs, that exploded in the Westfield ferry boat, July 30th, 1871, causing the death of about 100 persons and wounding as many more. The writer was employed to investigate the case for the officials upon whom the duty was legally and technically incumbent. It was found that the cause of the explosion was the extensive corrosion of one of the girth seams of the shell. The accident occurred when the pressure was about that ordinarily carried and considerably less than that at which the boiler had been tested but a short time before. The energy liberated was therefore about the same as would be calculated as above from the known dimensions and capacity of the boiler. The destruction of the boiler itself, its displacement, and the destruction of that part of the boat adjacent to it, were minor effects of the accident.†

A boiler of the return tubular class was tested to the bursting point, under steam, by Mr. F. B. Stevens, at Sandy Hook, November, 1871. The water was up to the water-line and the energy liberated was thus the full amount calculated. As then reported by the writer,‡ "when a pressure of 50 lb. was reached, a report was heard which was probably caused by the breaking of one or more braces, and at 53 1/2 lb., the boiler was seen to explode with terrible force. The whole enclosure was obscured by the vast masses of steam liberated; the air was dotted with the flying fragments, the largest of which, the steam drum, rising to a height variously estimated at from 200ft. to 400ft., fell at a distance of 450ft. from its original position. The sound of the explosion resembled that of a heavy cannon. The boiler was torn into many pieces, and comparatively few fell back upon their original position." This boiler had been tested by hydrostatic pressure, before its explosion, up to a pressure exceeding by 5 1/2 lb. that at which the explosion occurred.

The writer subsequently calculated the amount of total energy stored in this boiler and analysed the effects of the explosion, coming to the conclusions: § "(1) That it is very certain that the energy of this explosion, and all of its tremendous effects, were principally due to the simple expansion of a mass of steam suddenly liberated at a moderate pressure, by the general disruption of a boiler of very uniform but feeble strength. (2) That in this

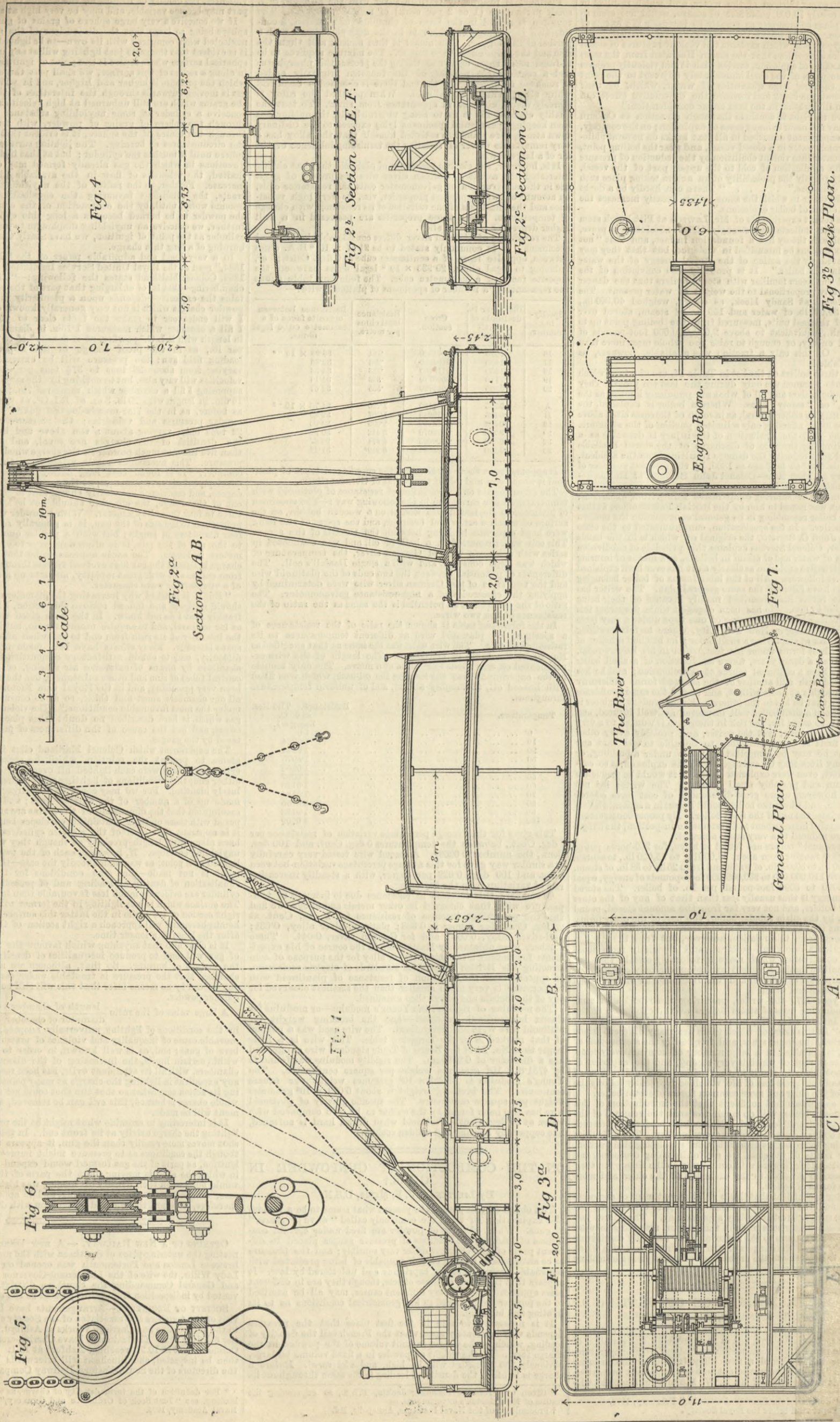
* The theoretical effect of good gunpowder is about 500 foot-tons per pound, according to Noble and Abel. † "Journal of the Franklin Institute," September, 1871. R. H. T. ‡ "Journal of the Franklin Institute," Jan., 1872. § "Journal of the Franklin Institute," Feb., 1872.

* Read before American Society of Mechanical Engineers.

* The Locomotive, September, 1884.

FORTY-TON FLOATING CRANE, STETTIN HARBOUR.

(For description see page 7.)



40-TON FLOATING CRANE.

Fig. 8

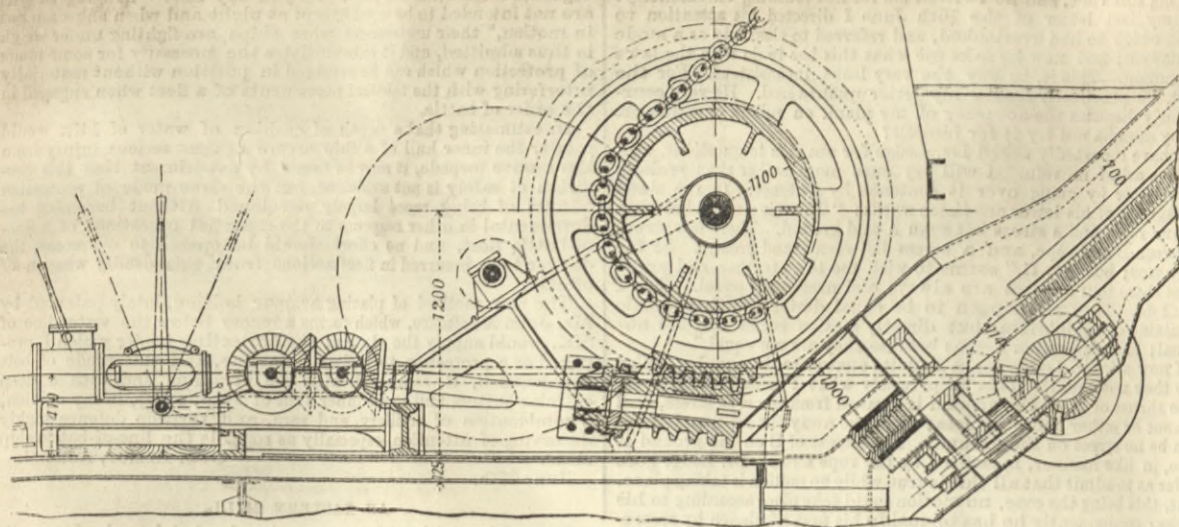
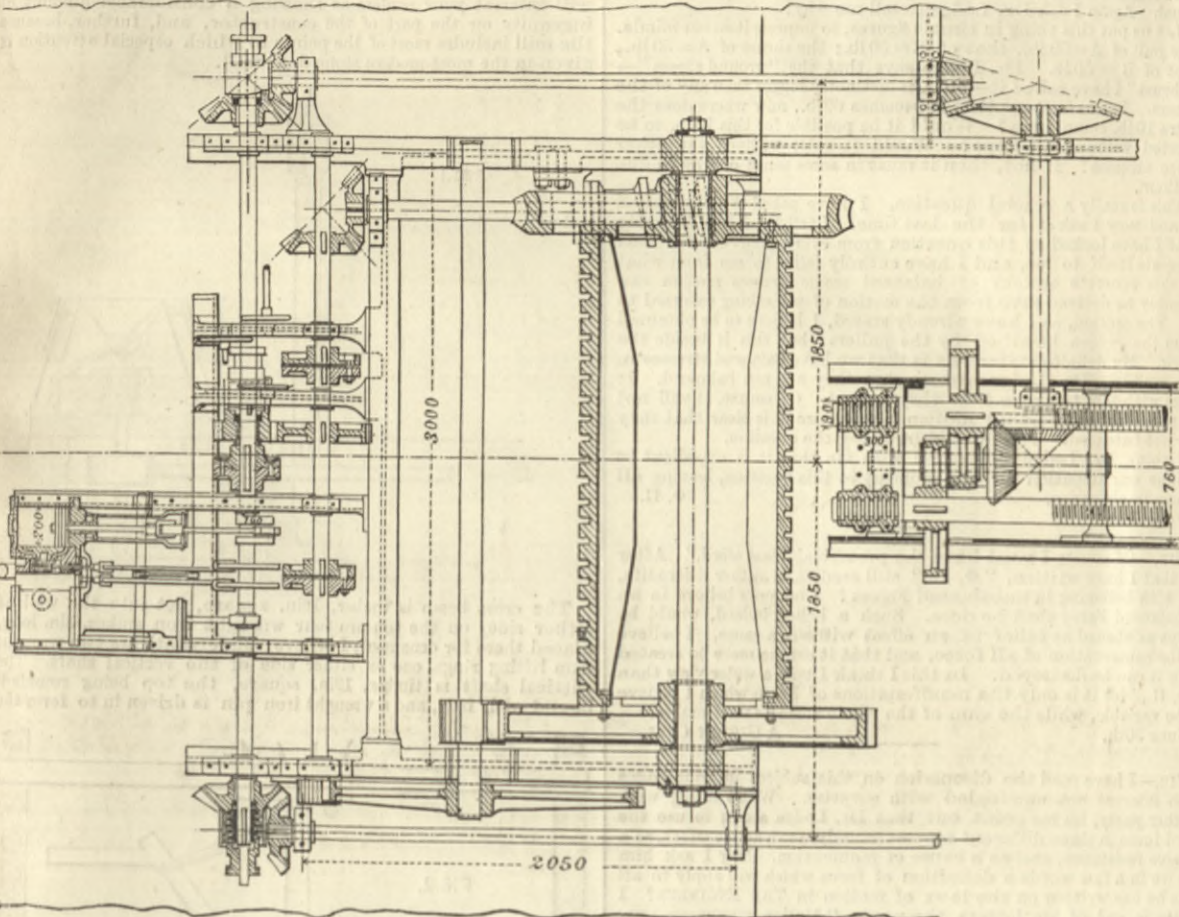


Fig. 9



We are indebted to "Glaser's Annalen für Gewerbe und Bauwesen" for the following description of a floating crane erected in 1883 at Stettin harbour. It was originally intended that the crane should be stationary, and fixed on the quay; but this idea was abandoned on account of difficulties expected in providing a substantial foundation, and it was decided to erect the crane on a pontoon floating in a basin specially constructed for it, as shown in Fig. 7, page 6. To facilitate the transfer of the loads between vessels and rail or road, one corner of the pontoon is pivotted to a centre pile, and it can swing round

and seven transverse girders which transmit the load to twenty-one I iron ribs placed longitudinally along the bottom. The pontoon is covered by a water-tight wooden deck, and at the rear end is fixed a ballast tank capable of holding 45 tons of water. The engines and boiler are placed in the hold of the pontoon at the rear end, whilst the sockets for the shear legs are placed at the front end immediately over the two main girders. These girders and three of the transverse girders are made use of for subdividing the hold into nine water-tight compartments.

FIG. 13.

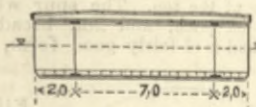


FIG. 12.



FIG. 11.

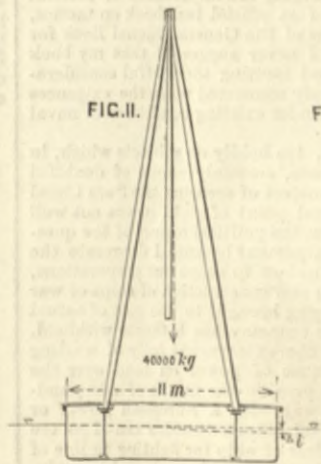
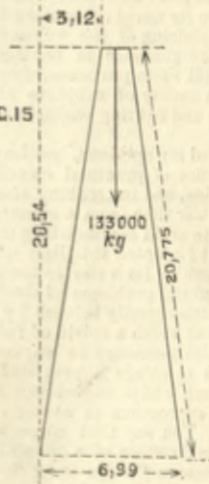


FIG. 15.



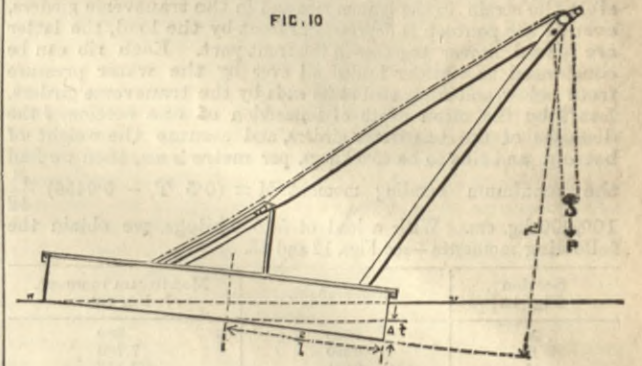
this point into any convenient position. Ordinarily the crane is to be used in this way, the vessels being brought up to it; and it is only in certain special cases that the crane will be required in other parts of the harbour. As such cases are not expected to be very frequent, the pontoon has not been provided with special propelling mechanism, and for locomotion it must rely on its own capstans or on a tug.

The motive power for all the operations necessary in loading and unloading vessels is furnished by a vertical boiler and a twin engine with reversing gear, no provision for hand power being made. The pontoon is 20m. long and 11m. wide, wholly made of iron, the plates at the bottom being 8 mm., and those at the sides 6.5 mm. thick. The weight of the crane and its mechanism is distributed over the bottom by a system of girders, consisting of two longitudinal girders placed 7m. apart,

As will be seen from Fig. 1, page 6, and Fig. 8, the derrick is movable, its lower end terminating in a cross-head, which, by means of two screws worked from the engine, can be shifted up or down between the inclined fixed guides. In this manner the crane legs can be inclined more or less, and the load shifted out or in. The maximum distance from the crane hook to the front edge of the pontoon is 8m. with a load of 14 tons. Each of the crane legs consists of four angle irons with diagonal bracing. The footsteps are cast iron bearings with top straps to prevent the crane legs lifting off; two tie rods prevent their being forced out laterally. The derrick consists also of four angle irons braced diagonally, and its head is hinged to the top of the crane legs by a turned bolt, which at the same time serves for the three top sheaves. There are three sheaves in the lower block, so that, abstracting from friction, the strain on

the chain is one-sixth of the load. The maximum load is 40 tons when the draught of the pontoon is 1750 mm. in front and 250 mm. at the back. With a load of 13 tons the pontoon is horizontal. As will be seen from Figs. 8 and 9, the chain drum is worked by spur gearing and by a worm and wheel, the two gears being connected by a coupling, having just sufficient play to allow the worm to lag when the load is being raised by means of the spur gear, and conversely, to allow the spur pinion to lag when the load is being lowered by means of the worm gear. In this manner the winch combines the safety of worm gearing without its waste of power through friction. The engine has cylinders of 200 mm. diameter and 200 mm. stroke—about 8in. by 8in.—and runs at 150 revolutions per

FIG. 10.

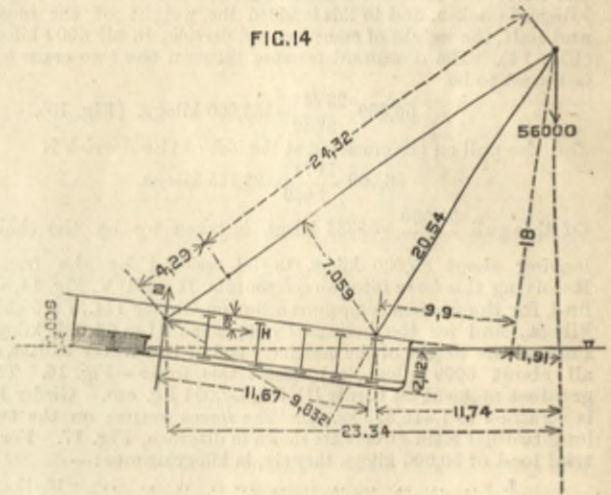


minute; normal steam pressure, 105 lb. The power is transmitted from the engine to the winch, the derrick screws, and the capstans, by means of friction clutches, and either of these may be worked singly, or two or more of them simultaneously, as required.

On the deck of the pontoon are fixed two rails, on which a railway truck or wagon can be placed, by drawing in the derrick until the crane head is vertically over the rails. It is thus possible to unship a whole railway wagon or small locomotive, and carry it to another part of the harbour to be landed.

The author of the article in *Glaser's Annalen*, Herr D. Blauel, from which we take our description, enters into some theoretical questions, and since his method of treatment appears to be both simple and practical, we add to our above description an abstracted translation of Herr Blauel's calculations.

FIG. 14.



A. *Stability.*—The depth of immersion of the pontoon with ballast tanks filled is, from actual measurement, 0.510 m. at the front edge, and 1.140 m. at the back edge; average, 0.825 m., which corresponds to a total displacement of 181,500 kilogs. The amount Δt (see Fig. 10) by which the draught in front exceeds and the draught behind falls short of the mean draught can be calculated with sufficient approximation by the formula—

$$P \left(18 + \frac{2h \Delta t}{l} \right) = 1000 \cdot 11 \cdot 10 \Delta t \frac{2 \cdot 10}{3} + M,$$

where P is the load suspended on the crane; l the length of the pontoon; h the height of top sheave above bottom of the pon

FIG. 16.

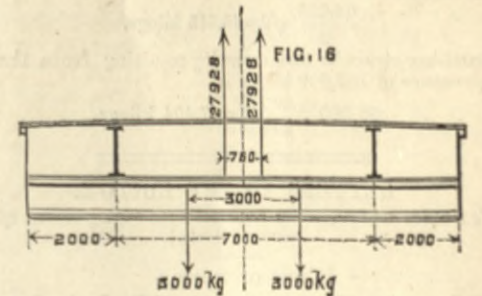
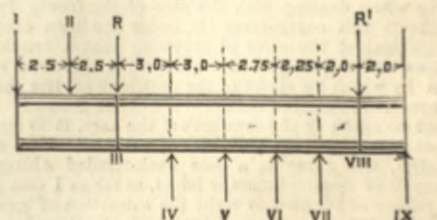


FIG. 17.



toon; and M the static moment of the crane without load in reference to the transverse axis of pontoon. Since for P = 0, trial has shown that Δt is 0.315, we obtain by substitution into above formula—

$$M = 231,000 \text{ kgm.}$$

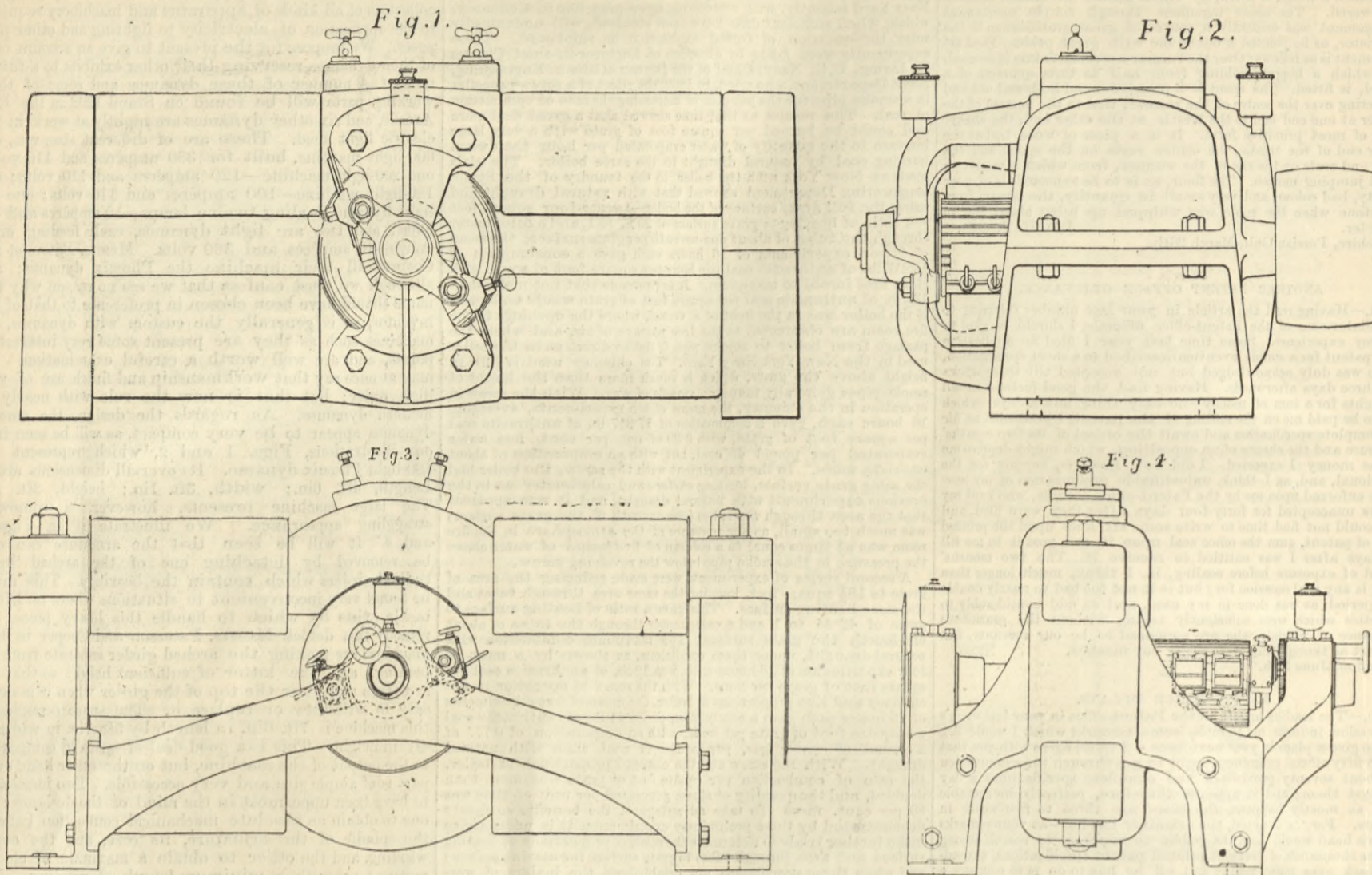
By substituting the value for h = 20.5 m. and l = 20 m., we obtain for a load of P = 50,000 kilogs.—

$$\Delta t = 1.060 \text{ m.,}$$

to which corresponds in front a draught of 2.112 m., and at the back a draught of -0.008 m. Both these measurements were verified at the trial. With a load of 40,000 kilogs., $\Delta t = 0.751$ m. and the draughts are 1.758 m. and 0.236 m. respectively.

B. *Stability against wind pressure.*—It is assumed that the pressure of wind be 125 kilogs. per square metre of exposed gross surface, no deduction being made for the openings between the bracing rods on crane legs and derrick. If a load of

MESSRS. PATTERSON AND COOPERS DYNAMO-ELECTRIC MACHINES.



the core makes it necessary that the cross section of the latter should be rectangular, it is evidently best to choose that rectangle which gives a maximum of area with a minimum of outline, and in this respect comes nearest the circle. If the assumption with which we started be correct, we come to the conclusion that the core should be of square cross section, and Messrs. Paterson and Cooper's armature would have to be only 4½ in. wide if its present radial dimensions were maintained. But our assumption is not correct, or, rather, it is impracticable to saturate a core of such form, for it would require too large an expenditure of energy to excite the field magnets sufficiently. We may regard the armature of Messrs. Paterson and Cooper as a compromise between what is theoretically right on the one hand and commercially expedient on the other hand. The internal diameter of the armature is 11 in., and the radial depth measured from the inner circumference to the bottom of the grooves 4½ in., giving a cross sectional area of 40.5 square inches. The conductor wound on the armature consists of a cable containing fifty strands .048 in. wire, having a total area of .091 square inches. A cable is chosen in preference to a solid wire on account of being more flexible. The perimeter of the winding is about 30 in., and since there are 2 × 84 = 168 turns on the armature, we find the total length of wire to be 140 yards. This gives 1.27 yards for every volt in the external circuit, an exceptionally good performance. The calculated resistance of the armature is about 0.013 ohms when working, and the loss of potential in the armature is with a full current of 380 ampères—5 volts. The density of current in the armature wires is 2100 ampères per square inch, that is exactly the same figure as in the Crompton machine described in our last article. The armature runs at 500 revolutions a minute, which corresponds to a speed of 2900 ft. at the circumference.

It has been pointed out in the beginning of these articles that armatures of the Pacinotti type are better adapted to be used with cast iron field-magnets than armatures with a smooth core. We find, indeed, that some of the smaller machines exhibited by Messrs. Paterson and Cooper have cast iron magnets—see our illustrations, Figs. 1 and 2—but the magnets of the large machine above described are made of wrought iron. They are 7 ft. 6 in. long, 9 in. square, and arched out in the middle for the reception of the armature. At the ends they are bolted to cast iron yokes, which form part of the framework of the machine. There are four exciting coils, each 22 in. long, and containing about 1½ in. of copper wire, viz., two layers of main wire, .360 in. diameter, and five layers of shunt wire, .102 in. diameter, the proportion of main to shunt being such that the electro-motive force increases slightly with the current in order that the machine may keep a constant electro-motive force at the far end of its main circuit. We shall give a longitudinal section through the machine in our next issue, as also some characteristic curves. The field magnet coils are not wound on metal formers as is the case with some dynamos of other makers, but are firmly bound together by canvas and tape, and the ends are protected by neat gun-metal flanges screwed to the body of the magnets. We have said in one of the former articles that the Pacinotti form of core is advantageous because the projecting teeth can be made to approach very closely to the

polar cavities. There is, however, a limit to this, as in practice it is found that if the teeth approach too closely the two corners of the pole pieces where the teeth leave them become hot. This is due to the magnetic reaction of the armature on the field magnets, whereby currents are set up in the metal of the pole pieces. Any kind of heating in a machine is coupled with a sacrifice of mechanical energy, and to minimise the loss thus occasioned the pole pieces in some of the Phoenix dynamos are subdivided by narrow slots running at right angles to the armature spindle. This can easily be done in machines with cast iron magnets, but in the large wrought iron dynamos heating is avoided by allowing rather more clearance between the extremity of the teeth and the polar surface than would be necessary on purely mechanical grounds. The magnetic resistance of the air space is thereby somewhat increased, and a slightly greater exciting power than would otherwise be necessary has to be applied. In the machine under consideration the initial exciting power of

end faces of the pole pieces in a manner similar to the original Bürgin and Weston machines. The magnets are straight bars of square cross section, and are provided with cast iron pole pieces bolted on. These pole pieces are subdivided by slots, as described above. The two machines in conjunction with the 180-light dynamo are employed for lighting the dining halls and the buffet by means of Bernstein lamps of 50-candle power each, whilst the arc machines supply current for twenty-four Phoenix arc lamps in the West Annexe.

PROPELLER AND CRANK SHAFTS.

THE quality most required in the material for propeller and crank shafts is combined toughness and strength, and judging from the following tabular report by Messrs. Kirkaldy and Son on iron made by the Mersey Forge for these purposes, that company has succeeded in obtaining this quality in iron for large shafts:—

Mechanical Properties of Three Pieces of Mersey Forge Hammered Iron Bar.

Test No.	Description.	Original.		Stress.		Ratio of elastic to ultimate.	Contraction of area at fracture.	Stress per square inch of fractured area.	Extension, set in 5 in.		Appearance of fracture.		
		Dia.	Area.	Elastic per square inch.	Ultimate per square inch.				at 40,000 lb. pr. sq. in.	Ultimate			
8	"S S" Hammered bar "A,"	inch	sq. in.	lb.	tons.	lb.	tons.	pr. cent.	pr. cent.	lb.	per cent.	per cent.	Fibrous
3288	2½ by 2½, rough turned 1½.	1.567	2.000	25,600	50,735	50.4	46.2	94,391	6.62	38.2			
3289	do. B	do.	do.	26,200	50,105	52.3	44.4	90,117	6.36	35.6			
3290	do. C	do.	do.	26,700	51,760	51.6	40.6	86,296	6.02	33.8			
	2½ by 2½ Mean			26,167 = 11.7	50,866 = 22.7	51.4	43.7	90,268	6.33	35.9			

the shunt coils on one half of the machine is from 11,000 to 12,000 ampèrereturns, whilst the exciting power of the main coils is about 20,000 ampèrereturns when the full current is flowing. The resistance of each main coil is .036 ohm, and as the four coils are coupled parallel their total resistance is .009 ohm. The resistance of the shunt is 17.7 ohms, and the current in the shunt wire therefore $\frac{110}{17.7} = 6.2$ ampères. The electrical efficiency calculated from these figures is $\frac{380 \times 110}{386.2 \times 118.6} = 91.25$ per cent.

The density of current in the main wire is 900 ampères to the square inch, and that in the shunt is 800 ampères to the square inch. In consequence of this low density and the shape of the coils, which have a comparatively large surface exposed to the air, there is very little heating. The commutator is of exceptionally large size, being 9 in. in diameter and 10 in. long. The segments have a wearing depth of 3 in., and the current is taken off by a double set of brushes, which are not only capable of adjustment round the commutator by the spur gear, as shown in our illustration, but which can also be altered in position relatively to each other by screws actuating sliding blocks. There is, however, so very little sparking and such a wide neutral zone in this machine, that we are inclined to think this second adjustment an unnecessary refinement.

The 200-light machine is similar in character to that just described, but differs from it in some details of mechanical construction. Thus the arched girders are omitted, and the bearings are supported in gun-metal brackets bolted to the

The iron is made from cold blast and all-mine pig irons, and for marine crank shafts and screw shafts the company claims that it possesses great advantages over best selected scrap, as it is free from impurities and is uniform in composition, while it asserts that long experience proves it to be superior to steel in that it is not so subject to sudden fracture. Its structural value as against torsional stress is also very high, specimens 2 ft. long and 2 square inches area, rarely fracturing until twisted between five and six complete turns, the fracture being always fibrous. Its tensile strength averages 25 tons per square inch; its ductility 37 per cent. in bars of 2 square inches sectional area by 5 in. long. With such a material there should be no difficulty in obtaining trustworthy shafts, even though a massive forging, has never the strength of a specimen cut from it.

THE DEESIDE RAILWAY—RECONSTRUCTION OF BRIDGES.—We understand that the directors of the Great North of Scotland Railway Company have resolved to reconstruct the bridges on their Deeside line, and have given the contract to Messrs. Blaikie Brothers, Footdee. The bridges are fourteen in number, and they are to be provided with steel instead of cast iron girders.

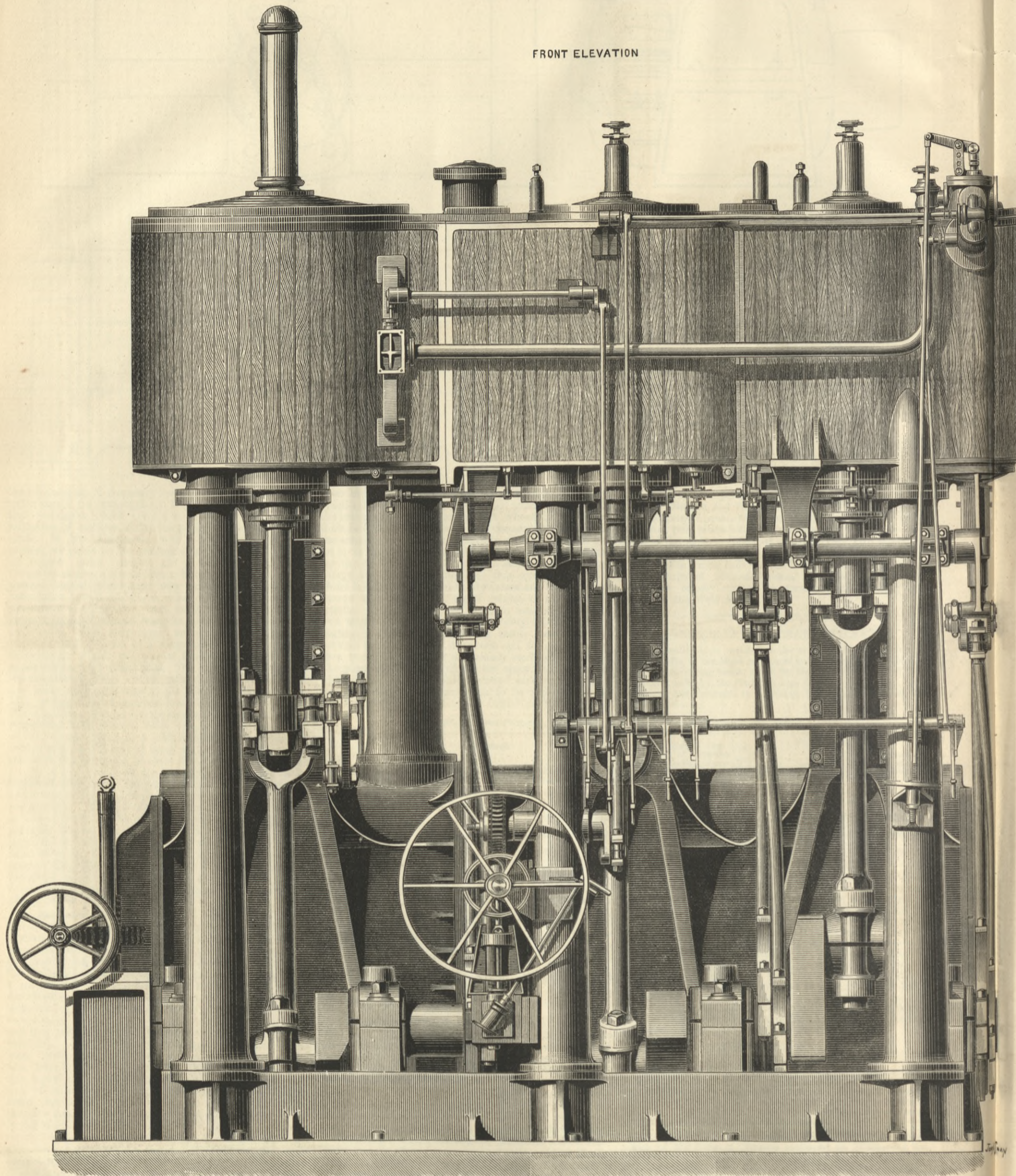
THE MANCHESTER SHIP CANAL.—Mr. Forster's Select Committee of the House of Commons continued on Wednesday the further evidence in support of the engineering case of the promoters. Mr. David Cunningham, engineer to the Tay Harbour Trustees, said the works done on the Tay bore a striking resemblance to what was proposed on the Mersey, and he believed, if the proposed works were carried out, they would prove rather beneficial than otherwise to the Mersey. Mr. Giles, M.P., said the abstraction of tidal water from the estuary by the proposed canal would be insignificant, and would have no injurious effect upon the bar.

TRIPLE EXPANSION ENGINES OF THE S.S. EASTWOOD.

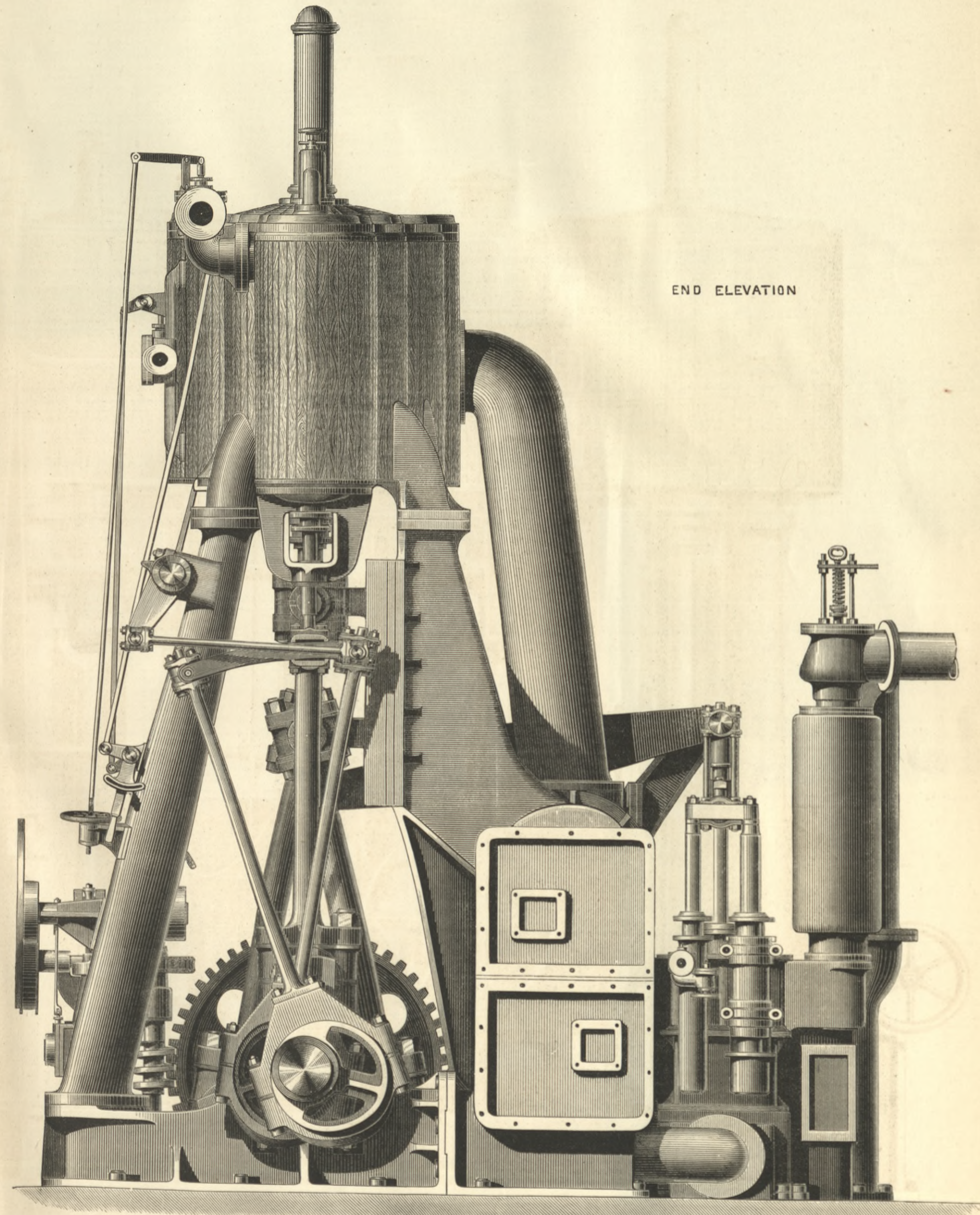
EARLE'S SHIPBUILDING AND ENGINEERING COMPANY, HULL, ENGINEERS.

(For description see page 11.)

FRONT ELEVATION

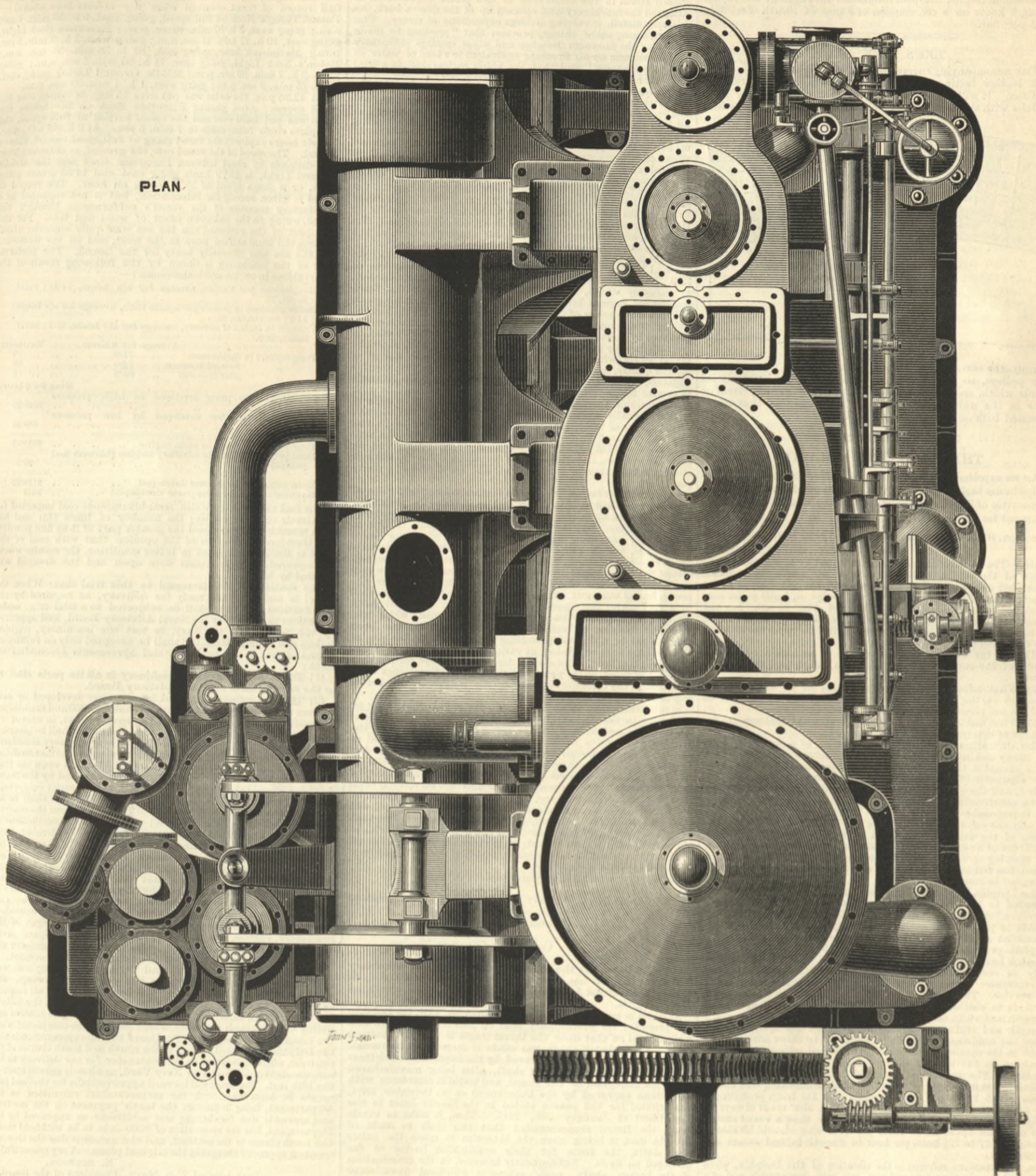


END ELEVATION



TRIPLE EXPANSION ENGINES OF THE S.S. EASTWOOD.

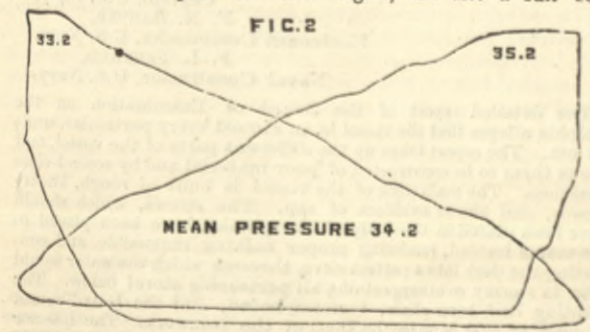
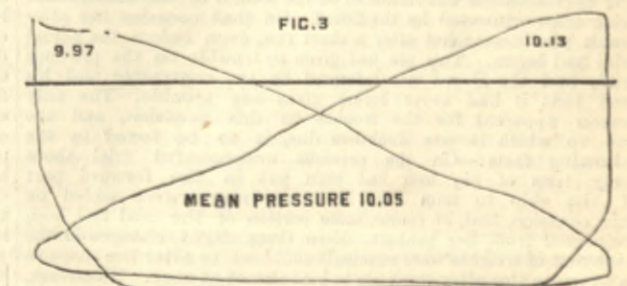
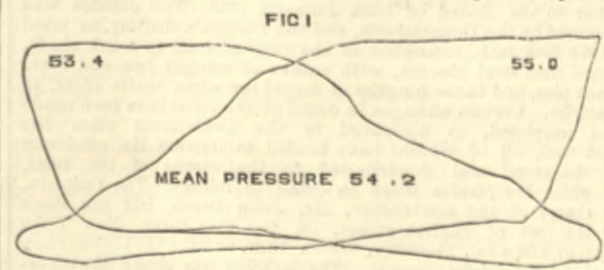
EARLE'S SHIPBUILDING AND ENGINEERING COMPANY, HULL, ENGINEERS.



We give this week a double page engraving of the triple expansion engines of the steamship Eastwood, built by Earle's Shipbuilding and Engineering Company, Hull. Above will be found a plan of this engine, and also a full set

of diagrams, which may be compared with those from the engines of the Isa published in our last impression. The cylinders of the Eastwood's engines are 19 1/2 in., 30 in., and 52 in. diameter, each having a stroke of 33 in. The piston-rods

are of steel fitted into cast steel crossheads, and secured to them by nuts. This arrangement admits of the pistons and rods being easily withdrawn at any time. The low-pressure piston



of diagrams, which may be compared with those from the engines of the Isa published in our last impression. The cylinders of the Eastwood's engines are 19 1/2 in., 30 in., and 52 in. diameter, each having a stroke of 33 in. The piston-rods

only has a tail-rod, and that instead of passing through a stuffing-box and being exposed, passes through a deep bush in the cylinder cover and is enclosed in a cast iron casing. The

condenser has a cooling surface of 1152 square feet. The air pump is of the usual kind. The circulating pump is a double-acting reciprocating pump fitted beside the air pump, and all the pumps are worked by means of a pair of levers from the low-pressure engine. Steam is supplied from one single-ended boiler 13ft. 9in. diameter by 10ft. 6in. long, made in accordance with Board of Trade rules for a working pressure of 142 lb. per square inch, and has a total heating surface of 1730 square feet. The furnaces are corrugated on Fox's plan, and made by Leeds Forge Company. This boiler we illustrated

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PARIS.—Madame BOYVEAU, Rue de la Banque.
 BERLIN.—ASHER and Co., 5, Unter den Linden.
 VIENNA.—Messrs. GEROLD and Co., Booksellers.
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 NEW YORK.—THE WILLMER and ROGERS NEWS COMPANY,
 31, Beekman-street.

PUBLISHER'S NOTICE.

* * With this week's number is issued as a Supplement, a Two-Page Engraving of the Triple Expansion Engines of the s.s. Eastwood. Every copy as issued by the Publisher contains this Supplement, and subscribers are requested to notify the fact should they not receive it.

TO CORRESPONDENTS.

* * All letters intended for insertion in THE ENGINEER, or containing questions, must be accompanied by the name and address of the writer, not necessarily for publication, but as a proof of good faith. No notice whatever will be taken of anonymous communications.
 * * We cannot undertake to return drawings or manuscripts; we must therefore request correspondents to keep copies.
 * * In order to avoid trouble and confusion, we find it necessary to inform correspondents that letters of inquiry addressed to the public, and intended for insertion in this column, must, in all cases, be accompanied by a large envelope legibly directed by the writer to himself, and bearing a 1d. postage stamp, in order that answers received by us may be forwarded to their destination. No notice will be taken of communications which do not comply with these instructions.

H. L.—Paddington to Bristol, 7ft. gauge.
 ENQUIRER.—(1) Two men ought to lift more than two tons. (2) Two men working at one good crane will lift more than two men at two cranes.

CORK-CUTTING MACHINERY.
 (To the Editor of The Engineer.)

SIR.—Can any of your readers inform me of the name and address of makers of machinery for slicing cork into strips, say about 4in. ?
 London, July 1st. J. W. H.

SUBSCRIPTIONS.

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Advertisements cannot be inserted unless Delivered before Six o'clock on Thursday Evening in each Week.

Letters relating to Advertisements and the Publishing Department of the paper are to be addressed to the Publisher, Mr. George Leopold Riche; all other letters to be addressed to the Editor of THE ENGINEER, 163, Strand.

MEETINGS NEXT WEEK.

SANITARY INSTITUTE OF GREAT BRITAIN.—The ninth anniversary meeting of the Institute will be held in the Lecture Theatre of the Royal Institution, Albemarle-street, on Thursday, July 9th, at 3 p.m. The chair will be taken by Sir John Lubbock, Bart., M.P., D.C.L., F.R.S. An address will be delivered by Professor W. H. Corfield, M.A., M.D., entitled, "The Water Supply of Ancient Roman Cities;" and the medals and certificates awarded to the successful exhibitors at the Exhibition held at Dublin in 1884, will be presented.

LONDON ASSOCIATION OF FOREMEN ENGINEERS AND DRAUGHTSMEN.—The half-yearly meeting will take place at the Cannon-street Hotel on Saturday, the 4th inst., at 7.30 p.m., when the auditors' report and Mr. James Irvine's offer to add £500 to the benevolent fund will be submitted to the meeting. Mr. W. T. Coates will then read a paper "On Technical Education from a Workshop Point of View."

THE ENGINEER.

JULY 3, 1885.

THE EFFICIENCY OF STEAM ENGINES.

DURING the last few years the action of steam in an engine has been investigated with a care, discrimination, and elaboration previously entirely unknown. The older school of engineers rested content with taking indicator diagrams for a few hours, weighing the coal fed into the boiler furnaces, and deducing the consumption of coal per horse per hour from the figures thus obtained. Messrs. Bryan, Donkin, and Co., were among the first to point out that something more than this was wanted, and they devised the exceedingly ingenious system of testing the efficiency of a condensing engine by measuring the feed-water and the temperature and quantity of the cooling water before and after its use. It is evident that the less the quantity of heat carried into the condenser per pound of feed-water pumped into the boiler the greater must be the efficiency of the engine, other things being equal; because the difference between the total heat sent into the engine and that rejected by it represents the heat converted into work. It is an unfortunate circumstance that the quantity of water required for condensing purposes is so large that it cannot be weighed, and must be measured by the discharge over a small weir. But hydraulic engineers have never yet made up their minds as to the proper method of measuring the discharge over a weir—that is to say, there is a given theoretical discharge for a given width and depth, but the practical discharge is always less than this. How much less is the

point at issue. No fewer than eleven recognised coefficients exist varying between '667 and '518. Thus, for example, let us suppose that the notch board or weir is 1ft. wide and the stream over it 2in. deep, then the discharge may be anything between 21'842 cubic feet and 13'477 cubic feet per minute. Of course there are influences at work which can be allowed for and which modify the coefficient. But after all is said and done, an element of uncertainty remains which has always militated against this method of estimating efficiency. Yet even if this were not so, it is evident that the work done in the condenser tells us nothing of what is going on in the engine; and it is in ascertaining this that recent workers have done more than any of their predecessors. The admirable paper by Mr. M. Longridge, the publication of which we commence this week, supplies an excellent example of what we mean. It will be seen that Mr. Longridge is able to follow the steam, step by step, as it progresses through the engine, and to show what is taking place inside the cylinders and valve chests. This close examination of what goes on in a steam engine possesses very great advantages, for it tells us, if we read the lesson it conveys aright, what is possible and what is not in steam engineering. But at the same time it brings us face to face with some apparently puzzling problems, which Mr. Macfarlane Gray will, perhaps, show us how to solve in his forthcoming book. To one or two of these we propose here to direct the attention of our readers.

We stated last week that in the high-pressure cylinder of the yacht Isa wet steam is always to be found, while dry steam exists in the low-pressure cylinder. The same circumstance has been noted with regard to compound engines of all kinds, but it seems to be more marked in triple-expansion than in other types of steam engine. Mr. Longridge finds that in the engines to which his report refers the exhaust from the high-pressure cylinders into the low-pressure consisted of 72'6 per cent. of steam, and 27'4 per cent. of water—that is to say, more than one-fourth of all the steam sent into the small cylinder of this compound engine was condensed. The quantity of water present is ascertained in the following way: The weight of feed-water pumped into the boiler being known, it is easy to ascertain how many pounds of feed-water are used per stroke. The pressure of the steam is ascertained from the indicator diagram; but it is known that the weight of steam sent into the cylinder per stroke would give a much higher terminal pressure than is actually obtained. The difference is due to the fact that all the feed-water passing into the cylinder in the shape of steam does not remain as steam, and anyone who knows the elements of arithmetic, and has a set of steam tables before him, can say at once how much steam and how much water are in the cylinder. The first question we have to ask ourselves is, Why was all this steam condensed? In the case of the engine cited by Mr. Longridge, there was used on the first day of his experiment 17'82 lb. of steam per horse per hour. Of this, 27 per cent., or 4'8 lb., was condensed. The engine indicated, in round numbers, 888-horse power; so that the two small cylinders actually condensed not less than 4262'4 lb. of steam every hour, or enough to work an engine up to 231-horse power, allowing 18 lb. of water per horse-power. To put this in another way, the two high-pressure cylinders condensed as much steam in every hour as would have been produced by 460 lb. of coal. A boiler with 23 square feet of grate surface and 450ft. of heating surface would have been required to make this steam. This represents a Cornish boiler of no small proportions. The two cylinders which did this condensing work were 26in. diameter and 6ft. stroke. Their cooling surface was, therefore, about 55 square feet each, allowing one cylinder coverface and one piston face. Thus it appears that 110 square feet of metal carefully clothed were as efficient in condensing steam as a surface condenser with 400 square feet of surface, over which 9 tons of water per hour are passed. Such figures as these seem to give incredible results, and we have here one of the puzzles which require solving. How is it possible for 110 square feet of surface, with a difference of temperature of only 91 deg. Fah., and that obtaining for only a portion of the time, to do the work? With this range the transfer of heat will take place at the rate of 233 units per square foot per hour per degree, or for 110ft., 2,322,330 units per hour; and taking the heat given up at 950 units per pound of steam condensed, this would suffice for the condensation of only about 2444 lb. per hour, or just about one-half the quantity actually condensed. Furthermore, it must not be forgotten that as the pressure fell some re-evaporation took place. How are we to account for the wonderful efficiency of these two small cylinders, with the limited range of expansion, in them as condensers? No surface condenser ever sent to sea can compare for a moment in efficiency with them. Of course, as we have said already, the facts are not peculiar to the engines tested by Mr. Longridge. As we pointed out last week, the high and intermediate pressure cylinders of the Isa are always wet, and this although they are carefully jacketed, and the condensation in the jacket is so great that the auxiliary feed has to be frequently put on to keep the water right in the boiler. Are we not justified in at least asking if we are not here face to face with conditions differing in some way and materially from those which obtain in an ordinary condenser?

Now let us see what happens to this 63 per cent. steam and 27 per cent. water when it gets into the low-pressure cylinder. At the end of the period of admission—that is to say, when the admission port of the low-pressure cylinder is closed—there was no less than 44'2 per cent. of water in it. As the pressure fell, however, during expansion some re-evaporation took place, and at the end of the stroke there was only 35 per cent. present. At one time—that is to say, when one-ninth of the whole period of expansion had been reached—no less than 48 per cent., or nearly one-half the whole quantity of steam had been condensed. The really marvellous aspect of all this is the power of condensation possessed by the comparatively small quantity of metal used, and it illustrates very clearly the absurdity of the argument that the use of a second cylinder in a compound engine tends to prevent

condensation by limiting the range of temperature. Some influence much more potent than any universally recognised is at work to bring about condensation in engine cylinders; and we begin to ask ourselves whether all that we learn from the indicator concerning condensation is quite true. To this, however, there can be but one reply—within the limits of error of the instrument it is true? What, however, are the limits? It was long since pointed out by Mr. D. K. Clark that wet steam loses its mobility to a very large extent; so much so that the back pressure in outside cylinder locomotives is, other things being equal, always greater than with inside cylinders. Now, if we have steam in a cylinder thoroughly wet, it is quite possible that it may produce a comparatively sluggish action in the indicator piston, and that the pressures might appear to be less than they really are. This is, however, merely a hint, the shadow of a suggestion. At all events, it is quite certain that no rational explanation of the great efficiency of a cylinder as a condenser of steam has yet been put in print.

Another puzzling circumstance about Mr. Longridge's engines is, that the results obtained from them were erratic. The cylinders were sometimes much more efficient as condensers than at other times. Why this should be so it is impossible to say. It will be seen, too, further on, that the results obtained with the engine on the second day were so far anomalous that Mr. Longridge has to defend his diagrams against possible charges of inaccuracy. We shall, however, postpone our consideration of this point until our readers have gained some idea from the report itself, of the nature of the valuable information it is intended to convey.

BOILERS WITH FORCED CIRCULATION.

In another page will be found a notice of the Stiletto, a yacht for which it is claimed that she is the fastest steamboat in the world. This claim is based on the assertion that she ran thirty miles in seventy-seven minutes—23'7 miles per hour. This is tremendous steaming, no doubt; but we are not told anything about currents, or tides, or winds, and it is fair to presume that these were not against her. The power required to propel ships increases in an enormous ratio with the speed, and the Stiletto has, perhaps, more power in her than any other craft afloat. She has been built by the Herreshoff Company, and steam is supplied by a Herreshoff boiler. This boiler consists of coils of tubing, through which water is forced by a circulating pump. If we are to believe all that we are told, Mr. Herreshoff has succeeded in overcoming difficulties which at one time threatened to baffle him in his pursuit of a forced circulation boiler. In this country various attempts have been made in the same direction, but they have all ended in what may be termed a qualified failure; that is to say, a forced circulation boiler can be made which sometimes works very well indeed, but it can never be depended upon, and requires under all circumstances to be carefully watched by highly competent men; if not, the coils will be burned or the boiler will men.

The idea of forcing water through a coil of tubing over, or rather in, a furnace, is extremely seductive. Any pressure can be carried with safety—in theory, at least. The weight of the steam generator is on the whole small, because, among other reasons, the quantity of water in the boiler is very small. The surfaces are very efficient in the production of steam, and the risks of furring up on the one hand, and of corrosion on the other, are reduced to a minimum. Repairs, too, can be easily effected, and there is little or no chance of explosion. Unfortunately, however, there are grave drawbacks to the system. The pump valves are prone to give trouble. Gun-metal and brass valves will not stand the high temperature. They become brittle, and break to pieces. A similar difficulty is not encountered in ordinary feed pumps, because the feed-water is seldom, if ever, hotter than 250 deg.; but steam of 185 lb. pressure has a temperature of 382 deg. It has been found that nothing will stand but steel. This difficulty got over, there remains one hitherto unconquered. It is all but impossible to tell how much water is in the boiler. There is attached to the coil a cylinder, or receiver, in which the steam is separated from the water. The steam pipe to the engine is attached to the top of this; the suction pipe of the circulating pump is fixed to the bottom. This receiver may be three or four feet high, and the water level will rise and fall in it a couple of feet in half as many minutes. Then it will disappear altogether, and the moment afterwards the receiver will fill to the top with a rush. If the engine ran perfectly steadily, and the fire never varied in intensity, all would go well perhaps, but the smallest difference in either, or both, is felt at once by the water in the boiler, with the results stated. Much trouble, too, is often experienced in the matter of working the circulating pump. This pump has practically no load on it, but it has not always solid water to pump, and incessant vigilance is needed to ascertain whether it is, or is not, at work. When the furnace is urged, as it is in the case of the Stiletto, by a fan, a failure for a few seconds in the action of the pump will lead to one of the coils becoming red-hot, and very probably bursting. This calamity has often occurred to the Herreshoff boiler.

Seeing, however, that there is distinctly a field open for the use of forced circulation boilers, those who have some time and capital to spare might spend both to worse advantage than in an endeavour to make the system a success. The first thing to be done is to devise some means by which the position of the water level in the boiler may be ascertained. One of the great defects of such boilers as hitherto made appears to us to be that the length of tube employed has been excessive. It does not require much penetration to see that if we put a coil of 50ft. or so of 2in. tubing in a furnace, the steam made in the lower portions next the hottest part of the furnace will in all probability blow the water out of the top part; and this action is no doubt one reason why the reservoir to which we have alluded fills and empties suddenly in the way stated. The circulating pump, after working badly for a few strokes, suddenly catches hold and sends into the coil a lot of water of comparatively low temperature,

This checks ebullition for a moment, the water next becomes heated suddenly, a portion of it flashes into steam, blows all the water out of the tube before it, and fills the reservoir with a rush. At all times this species of action must be going on, more or less, in long tubes. In a short tube, on the contrary, the action may be very steady. Of course the difficulty of making one pump keep up a uniform circulation in a number of short tubes is considerable. To borrow a simile from the electricians, with a long tube the pump works in series; with several short tubes, in multiple arc. It would appear, however, to be quite possible to provide a slide valve with several ports in it, so that the discharge from the pump would be divided among any reasonable number of tubes. Thus, say, there were six, then port one would be open for the first sixth of the stroke, port two for the second sixth, and so on. It may be suggested that the same result would be got by putting in resistance, so that the delivery from the pump must be distributed. We are not quite clear, however, that this arrangement would work. The construction of the external furnace casing again is a much more important factor in the whole matter than appears at first sight. It may easily be so heavy and expensive that all the advantages of forced circulation boilers may be lost. Indeed, this has already been found to be the case in more than one instance. The method which most commends itself to us is to use a coil of piping for the purpose. As it would be practically shielded from the fiercest heat of the furnace, the necessarily great length of it would not be objectionable. The rest of the heating surface would, of course, be made up of short lengths of tubing.

We have omitted to mention one of the great advantages of the forced circulation boiler. It is that steam may be got up in a few minutes, and this cannot be over-rated for naval warfare. It is further to be borne in mind that bullets which might cause a disastrous explosion in the case of an ordinary torpedo boat boiler might do so little injury to the pipe boiler that the lives of a crew would not be lost. It is by no means unlikely that a good deal more may be heard of this type of boiler, and it certainly deserves more consideration than it has yet received.

THE BLOCK IN THE SUEZ CANAL.

THE ENORMOUS inconvenience and delay, to say nothing of the serious pecuniary loss which they have entailed, arising from the recent sinking of a dredger in the Suez Canal, have sufficiently revealed to us one disability arising from the conduct of a large carrying trade through an artificial channel. It is impossible not to realise from the late accident how dependent are British interests in the East upon an immunity which cannot, it is certain, be depended upon. Over and over again have correspondents called attention in the public journals to the results which might be apprehended should such a block as has recently been experienced occur during a hostile crisis, whether due to accident or design; but perhaps never during the history of the canal has such an apprehension received more practical illustration than during the last few weeks, when, by the sinking of one of the dredgers employed upon it, the whole of the vessels both outward and homeward bound through it have been delayed for fully three weeks. Now, unless we are prepared so to widen this canal, that, instead of being what it is, it is rendered almost a navigable strait, there is reason to fear that such an obstruction is liable to recur at any moment. It is scarcely within the range of possibility that such an enlargement of this great maritime highway can take place within our own time, though it would be rash to predict that the necessities of the future may not bring it about. We have, therefore, to consider what may best be done to obviate as far as possible the recurrence of an incident which has been productive of the great inconvenience and loss to which our shipowners and others have been subjected. Naturally the management of the Suez Canal would be but too glad to adopt any means within its power by which so desirable an end could be attained; and although those means appear to us, it must be confessed, to be exceedingly limited, it would yet seem as if some simple precautions might be taken which would at least largely reduce the chances of accident. Of course, in the navigation of a canal, even under the most stringent regulations, the risks of collision between vessels are many. These become extended when one of the objects of such possible collision possesses no mobility, such as might enable the captain of one of two vessels meeting in the canal to make up, by the movement of his own ship, for *laches* on the part of another. The presence of a considerable number of dredging machines must always be necessary on a waterway made through light and drifting sand, and the fact that such machines are moored, and therefore exposed to more than ordinary liability to accidents, should lead to special care in their construction, so as to avoid as far as may be possible their being sunk by contact with a moving vessel. There are few of the steamers, we should say, which pass through the Suez Canal which are not free from such a liability. Scarcely one of these leaves the British shores, at all events, which is not built in water-tight compartments, and is therefore exempt from the certainty of sinking in a hurry, which must attend those not furnished with similar safeguards. Even supposing one of their compartments to fill, the worst effect to be expected would be grounding, owing to increased immersion; and a very few hours of pumping in such a case would suffice to float a vessel again sufficiently to enable her to be drawn out of the highway and so leave a free passage. It would certainly appear as if the directors of the Suez Canal Company would be justified in forbidding the use of their waterway to any ship not provided with these water-tight compartments. A notice of but a few months would suffice to prevent such a restriction bearing with too great hardship upon shipowners, the great mass of whom would certainly have no reason to complain were such a restriction at once imposed.

But the freedom from risk enjoyed by vessels built after the plan indicated has not, we believe, been insured by a similar design being applied to the building of dredging machines. Nor do the form or the requirements of these

last lend themselves freely to the adoption of the system of watertight compartments in sufficient number to insure floatation when seriously injured. In the case of dredgers which have their ladders placed amidships, and operate through a longitudinal well which practically cuts the vessel into four distinct parts, it is the centre sections which possess the least amount of buoyancy, and it is manifest that the want of floatation power is thereby largely increased. Much remains yet to be done in the improvement of craft of this class, in spite of much that has of late years been accomplished towards this end, and one most desirable form of such improvement must manifestly be the increase of protection against sinking in the event of collision. For, as we have above pointed out, the master of a dredger is helpless to direct his vessel so as to avoid, by skilful steering, the delivery of a blow by a colliding vessel on the most vital part of his machine. He must remain passive to be struck wherever chance or the ignorance or carelessness of his opponent may direct it.

It occurs to us that, admitting the difficulty of so constructing dredgers in water-tight compartments in sufficient number to ensure vitality to such as remain uninjured, something may yet be done to minimise the consequence of sinking in a narrow and shallow waterway like the Suez Canal. Improve it as we may, a dredging machine must always remain one of exceptional weight and cumbersome, and any form to be given it must, we fear, still possess considerable elements of weakness owing to those conditions. It would seem, indeed, that any ordinary lifting gear applicable in a canal, with its restricted width, would fail to raise such a vessel. That such a fear is well based may, we hold, be assumed upon the fact that it has been found necessary to remove that now lying in the bed of the Suez waterway by blowing her to pieces by dynamite. It is certainly scarcely creditable to modern engineering science that no alternative method for clearing such an obstruction could be adopted. It does not seem to us to be impracticable that dredgers could be built in sections capable of being readily detached by divers. These, if united by screw bolts instead of by rivets, and strengthened over all by girders readily com- at-able, might be detached, we should say, without much difficulty by divers, and such sections, when once separated, could be raised by means which we have said do not appear to us to be applicable under the circumstances to the weighty machine as a whole. We have before directed the attention of our readers to the need of improvement in machines of this class. It did not occur to us when thus doing so to refer to the special form of such improvement which the late block in the Suez Canal has shown to be needed. We trust it may engage the attention of the many skilled engineers who devote themselves to the building of dredgers.

THE RICHMOND WATER SUPPLY.

PUZZLES in water supply still worry the minds of the Richmond Town Council. On the 23rd ult. a local government enquiry was held respecting a proposed loan for water supply works. Mr. Peirce, resident engineer, has proposed, as mentioned in our last impression, that a well already made should be deepened, with a view to increasing the supply. A question arises which bears relation not only to the condition, homogeneous or shattered, of the chalk in which the well is made. It has been found that when the pumps have been stopped, the water has risen rapidly at first, gradually slackening in the speed of rising as it neared the surface. From observation Mr. Peirce has concluded that "the supply doubles itself per minute at every 15ft. down." From this it appears that a calculation has been made purporting to show that five and a-half times the present rate of inflow will be obtained by deepening the well 45ft. Whether there is any practical fallacy in this deduction will, of course, depend on the dip of the strata in the locality and the condition of the chalk in the neighbourhood of the well; but it would be difficult without knowing these accurately to say at what depth the rate of increase began as rapidly to diminish as the inflow does as the water nears the surface when the pumps are stopped. When the water has reached this hydraulic level there is no unbalanced head. The rate of inflow will increase in a rapid ratio with the disturbance of this balance by lowering the water surface; but at what depth this increase will again diminish can only be surmised, as it will in any case be affected by the depth at which the main influx resides. The surmise may, of course, be based upon more information than the observation above mentioned; but on that alone it would be difficult to promise more than a high probability of a very considerable increase. As already mentioned, it has been decided to expend £800 in trying the experiment, and in the course of the inquiry above mentioned Mr. Henry Davy was asked if he could supply the connection between the rate of the well filling up and the rate at which the water was drawn off. In reply, he said "he thought a test ought to be made with the pumps going. If the present rate was then maintained for twenty-four hours, there would be good ground for going deeper. He supported Mr. Peirce's proposal on the ground that a small expenditure of £800 would enable them to discover what was the actual yield of the upper chalk through which the bore holes went. The bleeding surface of the chalk was 360 square feet. If they found by putting down experimental pumping gear into the bore hole a considerable increase of the water by lowering the water level 100ft., then they might know that the chalk would yield a considerable quantity of water, and they would be justified in carrying out Mr. Peirce's scheme and lowering the dummy well and driving adits. That was the only way they would obtain a larger supply. The present well yielded in round numbers 116,000 gallons in twenty-four hours through the dummy well. The bore-hole also yielded 15,000 gallons, but that was now cut off. If they opened the bore-hole, they got 131,000 gallons in twenty-four hours. The requirements were 463,000; or, omitting street watering, 441,000 gallons. So that if the present well be made to yield 331,000 gallons in addition to what it pumped at present, the supply of the town would be met. The required increase would be got at 844ft. from the surface, according to the plans before them; but it was proposed to put an experimental pump 233ft. from the surface, so that a very large discount might be taken from the figures before them to get at a justification for trying this experiment. He was of opinion that the increased quantity obtained would be sufficient to justify them in expending £800, and they would then have data to go upon in regard to a larger scheme. "The water supply question would then be put beyond doubt altogether." The question is not quite so simple as it

looks, and the chalk supply is not so calculable a quantity as might be supposed. Why does not Richmond go to the Thames for its water?

TERMINAL CHARGES AND THE TRADERS.

CHAOS has come again. Traders must either carry out in earnest their projects for making the inland waters and canals more than at present means of competing with the railway companies, or they must induce Parliament to come to their aid. The new trouble has arisen out of the reversal, to which reference is made in our Birmingham letter, by Mr. Justice Manisty and Mr. Justice Wills, of the ruling of the Railway Commissioners, that terminal, siding, and certain other charges made by the railway companies, as carriers, really come within their duty as a railway company. Their lordships laid down that the duty of the Commissioners is to fix what are reasonable carriers' charges in excess of the maximum railway rates, and in doing so, to see that the line between the conveyance of goods and the other services by the company was strictly kept. The precise case upon which the Queen's Bench has ruled will now have to go back to the Commissioners, and until they have decided, their lordships do not enter upon the second case, which involves an appeal under the same class of decision by the Commissioners. The traders had thought that not only had they been freed by the Commissioners' decision from responsibility to the railway companies for terminal, siding, and some other charges held by the Commissioners to be included within the maximum rates, but they had likewise hoped that the Commissioners' decisions could be made final. Their lordships, however, ruled otherwise; nor did they quite see why their own decision now given should not be subject to appeal. Obviously the question is one of very great importance to the traders, and the determined manner in which it has been fought by the railway companies indicates the serious view which they, too, take of it.

GOODS AND RAILWAY RATES.

THE application from Hull to which reference has been made in THE ENGINEER has raised the consideration of the question of railway rates for the carriage of goods to and from the seaboard in a new phase. It is a question whether the railway companies shall have power to define the rates charged, as long as they keep them below the legal maximum. The Hull instance supplies an illustration. The rates from and to Hull are not above the legal rate, and the point is, whether the railway company should have power to charge less than the rate to other districts. It is a thing of vast importance in these days of sea and land competition in carriage. If absolutely equal mileage rates were charged to all districts, the trade would perhaps not gravitate to Hull, but to ports which are nearer to the large centres of consumption of the imports and to the centres of production of the exports; and it is not a question which can be very easily settled off-hand. One of the very largest of the importing firms at Hull has publicly expressed its dissent from the movement on the ground that if it were there successful other districts would take up the principle, and apply it to the loss of that port in the end. There is this to be said for the present method in the case of Hull, that it has, on the whole, allowed the trade of that port to grow, and it does in some degree give to the consumer the privilege of choice of ports; but it has its disadvantages also, and thus it is not wise to pronounce a very dogmatic opinion offhand. The question will, however, have to be decided, not so much in the interests of one port, but in the interests of large areas of country; and if the trades of districts are best served by the choice of ports, and if the whole of the rates are below the legal maximum, it would probably be found to be most fitting that the question should not be decided on the application of one firm, but that some general legislation on sound principles, and probably based on a general reduction of rates for carriage, should be the result.

CROSSING REFUGES.

OF late years there has been a most desirable multiplication of those refuges which enable people to escape some of the dangers attendant upon crossing the most crowded of our London thoroughfares. We are aware that the authorities have shown themselves to be most ready to attend to representations made to them by private individuals as to necessities in this respect, and there are few of the most dangerous crossing places which have not been provided with a midway halting place. These serve not only as a safety haven for foot passengers, but are also a most efficient means of separating the passing streams of traffic. But the returns annually made by the police evidence to us how serious is the number of accidents to life and limb which still occur in the streets of London. Indeed, they appear from the constantly maintained annual average to be almost subject to some unknown law, and the fact would seem to indicate that there are causes operating to produce them which are not affected by additions yearly made to these midway halting places at our crossings. To one at least of such causes we can ourselves bear evidence. The London cabman is, as a rule, a thoroughly good man, demanding and receiving a large amount of our sympathy amid his needs and trials; but he is, at the same time, often most disregardful, in his anxiety to carry numerous fares, of those rules of the road the observance of which is necessary to the utility of the refuges. On most of these there are directions to keep to the right or left. We can only say that there are many places, and especially at the dangerous turning from St. James's-street into Pall-mall, where these directions are constantly disregarded. It is rare to see a policeman directing the traffic at that point, and until a few obstinate infringers of the rules have been fined, we fear we shall continue to have to see risks run daily at this and other exposed points.

LITERATURE.

New Formulas for the Loads and Deflections of Solid Beams and Girders. By WILLIAM DONALDSON, M.A., A.I.C.E. F.N. Spon.

IT is with much regret that we feel compelled to pass an adverse judgment upon Mr. Donaldson's little work "On Solid Girders," which, though small in bulk, is nevertheless prolific in the crop of egregious blunders and false assumptions scattered throughout its pages. The first of these appears on the top of page 3, where the author assumes that the internal stress—by which is meant the shearing stress—may be taken as uniformly distributed over the surface of the section. Now, although such an assumption is often made for practical purposes, it is none the less illicit to state it thus broadly as a first principle upon which to found such astounding statements as that "in any section between these two—that is, between the section of maximum bending moment and either end of the girder—

the acute angle, which the direction of the stress at any point between the neutral axis and the top or bottom of the beam makes with the vertical, increases as the distance of the point from the neutral axis." So it would seem our author innocently believes that the shearing stress does not vanish in the extreme fibres of the section. The above extract evidently involves this novel theory of the distribution of shearing stress; but, to remove all shadow of doubt, we have it explicitly stated on page 21 "that the intensity of stress per square inch, which tends to cause one fibre to slide over another in a longitudinal direction—such is the author's circumlocution for 'longitudinal slip'—must, like that of vertical sliding stress, be greatest in the extreme top and bottom fibres, and therefore the actual reactions between the fibres is of greater intensity than those between the fibres near the centre." These two passages, taken from the book, contain unmistakable evidence of what is nothing short of high treason and flagrant heresy against the laws of stress. For the distribution of shearing stress

per unit-area of cross section obeys the law $f = \frac{F}{I} \int y dy$,

in which expression F is the total shearing force at the section, and I the sectional moment of inertia relatively to the axis of flexure from which the ordinates y are measured. Wherefore, it will be seen, that the unit shearing stress f vanishes in the extreme fibres and attains a maximum at the neutral axis of the section. Be the section solid and rectangular, then we have $f = 0$ in the extreme fibres and $f = \frac{3F}{2A}$, a maximum, at the neutral axis. Hence in this particular case the maximum bears to the mean shearing stress $\frac{F}{A}$ the proportion of 3:2. Returning now to our

author's first statement, we find that he there promulgates a law not conformable to truth; because in the extreme fibres of the section the resultant stress, that is the stress compounded of the vertical and longitudinal component stresses, becomes in virtue of the equation $f = 0$ practically horizontal in direction; and therefore the angle, which its direction at any higher or lower point makes with the vertical, diminishes from 90 deg. to a fixed value at the neutral axis. We must therefore earnestly recommend our author to unlearn all his previous notions concerning the distribution of shearing stress and, as hastily as possible, to grasp firmly the idea that the vertical shearing stress per unit of sectional area vanishes in the vicinity of the extreme fibres, where the longitudinal component stress attains a maximum.

Being thus in possession of our author's notions about vertical stress, we are not at all surprised to find him stating that not only shearing stress, but also longitudinal slip, attains a maximum in the outer layers of the section. He has been led into this error by confounding longitudinal slip with longitudinal stress, and then presuming that the first must depend directly upon the second. It is easy to dispel this idea in a very few lines. Let T be the total longitudinal stress applied above or below the neutral axis of any section. Then the differential longitudinal stress tending to produce slip will be of the form $\frac{dT}{dx} = \frac{dT}{dM} \frac{dM}{dx}$. But

$\frac{dT}{dx} = F$, the shearing force at the section. Therefore, $\frac{dT}{dM} = F \cdot \frac{dT}{dM}$, where M is the local bending moment.

Now $\frac{dT}{dM}$ vanishes in the extreme fibres; hence also $\frac{dT}{dx}$, the stress producing slip, will vanish, and with it

all tendency to slide. In plainer and less technical phraseology, a beam in deflection manifests no tendency to ruffle its outer skin and creep after the fashion of earthworms. Whatever tendency there is to slide increases from the outer layers to the neutral surface, which sufficiently explains the ordinary practice of carpenters in tenoning composite beams along the line of connection of the planks. In a solid beam this joint is represented by the neutral surface, along which there exists the greatest tendency to slip.

Another point upon which our author imagines that he has thrown additional light is the fact that, to quote from the book, page 23, "the neutral axis does not necessarily in all materials ever (?) pass through the centre of gravity and does not maintain an invariable position, but that it is continuously changing its position with every change in the magnitude of the stress." Now it is perfectly true, and it was admitted long before Mr. Donaldson published his book, that the neutral axis of a series of forces varying as the distances of their points of application from this axis does not invariably coincide with a parallel line drawn through the centre of gravity of the surface. In this particular case of uniformly varying stress we know that the centre of application of the resultant lies upon the axis of y, or the axis conjugate of z in the central ellipse of inertia of the section, and at a distance from it expressed by $y_r = r^2 \div y_0$, where y_0 is the ordinate of the neutral axis and r the radius of gyration of the section relatively to the parallel axis passing through the centre of gravity. But when $y_r = \alpha$, we have $y_0 = 0$; that is to say, when the resultant stress becomes an indefinitely small and infinitely distant force, or in other terms, when the stresses acting at the section reduce to a couple or bending moment, the neutral axis coincides with the parallel axis through the centre of gravity. Now this happens to be the very case in point, and in fact it is the only case treated in the book before us; yet, strange to say, some infatuation has led the author to misapply the general principle, which, by the way, he never enunciates, and to extend it in its full force to the single exceptional case where it does not hold, and where the neutral axis invariably passes through the centre of gravity of the section. We are glad to find some indication of repentance on page 6, where, in order to establish equa-

tion (3), our author innocently, and probably unconsciously, begs the principle that the neutral axis passes through the centre of gravity of the section. On page 20 the reader will find an amusing illustration of the kind of transformation which a cross section is presumed to undergo during the process of deflection, which, if driven home, would lead us to believe that a box section, when deflected by vertical load, expands at the top and contracts at the base; so that the under-member or floor beam would be always in compression. We must leave with Mr. Donaldson the full responsibility of ratifying this extraordinary statement.

There are other amazing passages in the book which we cannot afford space to mention, and we must therefore conclude by pointing out a few of the clerical or mathematical errors scattered throughout its passages. We have on a former occasion, in the review of another work, mentioned that there cannot be "the three following," page 2, though there may be "the following three" assumptions. Presumably our author is not an Irishman "We will," instead of using "We shall," merely to express futurity or purpose. The square over the radical factor, top of page 12, should be omitted. The first factor, $W \left(\frac{x^3 - lx^4}{6 - \frac{lx^4}{4}} \right)$ in the

big bracket at the bottom of the same page should read $W \left(\frac{x^3 - lx^4}{6 - \frac{lx^4}{8}} \right)$, and as the two expressions agree in giving the same values for the limit taken—namely, $x = \frac{l}{2}$ —we are greatly tempted to suspect that the integration was adjusted by a fit. On page 19 the factor $\frac{5}{25}$ is given in repetition for $\frac{5}{24}$. In conclusion, we again regret our inability to say a good word for Mr. Donaldson's little work.

The formulæ which he proposes for deflection are new only in their greater complexity. Taking only one instance, if we had to choose between the ordinary formula, $y_0 = \frac{5}{384} \frac{wl^4}{EI}$ for the central deflection under uniform

load, and the new formula $y_0 = \frac{5w \left(1 + \sqrt{\frac{E}{E_i}} \right)^2 l^4}{128 d^3 b E}$, we

should decidedly declare in favour of the former and simpler expression, where the straightforward factor $\frac{1}{3EI}$ takes the place of the more complex co-efficient, $\frac{\left(1 + \sqrt{\frac{E}{E_i}} \right)^2}{d^3 b E}$.

From Keel to Truck: a Marine Dictionary in English, French, and German, for the Use of Shipowners, Builders, Barristers, Surveyors, Engineers, Naval Schools, &c. By H. PAASCH, K.C.A.R., Surveyor to Lloyd's Register. Antwerp: P. Ratinckx. 1885. Pp. 306.

THIS is a dictionary comprising, or intended to comprise, a translation in three languages of all the terms used in any part of the history of a ship, or water craft of any description. It is impossible to ascertain without some months' use of a dictionary of this kind whether it is complete or not, but as far as numerous tests can show it appears to be complete; and in order to make it more generally useful to those not possessing the technical knowledge which the author's thirty-five years' experience has given him, he has very fully illustrated the dictionary with shaded diagrams and drawings, illustrating ships and their parts and machinery and apparatus, the names of the details of all being given on the drawings, so that the user of the dictionary is aided by the eye in readily ascertaining the meaning of any term. The arrangement of the dictionary is excellent, and to be quite sure that the user shall not fail to find any term, the author has given an index in each of the three languages, an addition which is very acceptable, as so many terms are compound words, which make it difficult to find them without such an index. The book is divided into fourteen parts, the first of which is a descriptive list of the principal types of sea-going sailing vessels and steamers, with illustrations that make the descriptions easily comprehensible. The second part relates to wooden vessels and the wooden parts of composite hulls, and the different kinds of wood used in shipbuilding. Parts three and four deal with iron hulls and engines, boilers and machinery, apparatus, tools, and mechanical expressions. The illustrations of these parts are well done for their purpose, every part being clearly defined and labelled. Parts five to ten deal with anchors, chain cables, boats, capstans, pumps, windlasses, masts and spars, rigging, sails, tackles, and sundries, while the remaining parts deal with knots, bends, hitches, weights of materials, and appendix. The dictionary is one which will be useful to those who "stick to their desks and never go to sea" as well as to those who do go, and it appears to be as satisfactory in the terms relating to seamanship as in those of the structural or mechanical kind.

BOOKS RECEIVED.

Gas Engines. By William Macgregor. London: Symons and Co. 1885.

The Theory and Action of the Steam Engine. For Practical Men. By W. H. Northcott, C.E. Fourth edition. London: Cassell and Co. 1885.

Exterior Ballistics. By Captain James Inglis, United States Artillery School, Fort Monroe, Virginia. 1885.

Transactions of the Institute of Engineers and Shipbuilders in Scotland. Vol. xxvii., 1883-84. Glasgow: The Institute. 1884.

Transactions of the Institution of Naval Architects. Vol. xxvi. Edited by G. Holmes, Secretary. London: H. Sotheran and Co. 1885.

The Law Relating to Building, Building Leases, and Building Contracts, with a full Collection of Precedents with respect to matters connected with the Law relating to Building, with Notes and the Latest Cases under the various Sections. By Alfred Emden, Barrister. Second edition. London: Stevens and Haynes. 1885.

PRIVATE BILL LEGISLATION.

THE formation of a new Ministry has not interfered with the prosecution of the Select Committee's enquiries, but in one slight respect the most important of all the Committees this year has been directly affected. By the appointment of Mr. Dalrymple to a Junior Lordship of the Treasury, the Committee on the Manchester Ship Canal Bill in the Commons has been deprived of one of its most useful members. The circumstance was of course formally reported to the House, but instead of appointing another Member to fill the vacancy, the House authorised the remaining three members to continue the enquiry. Thus they avoided the necessity of a fresh member wading through the solid mass of evidence already taken, a process far less effective than hearing the witnesses, and neither side can feel aggrieved. The investigation is accordingly still dragging its slow length along, and promises to so continue for some time to come, despite the efforts of the Committee to shorten proceedings. Of general results of Committees, there is not much to be recorded since we last considered the subject. The London and Blackwall Railway Bill, for widening and generally improving the line from Fenchurch-street to Stepney, has been sanctioned by a House of Lords' Committee, their approval also authorising the company to raise £330,000 by shares, and to borrow £110,000. The same Committee have likewise passed the Bill for constructing tramways in King's Cross-road, Farringdon-road, Gray's Inn-road, and also, at the other end of the company's system, in Kentish Town-road, and Fleet-road, Hampstead. The Metropolitan Railway Company opposed the scheme, their ostensible reason being that it was contrary to the established principle to admit these tramways within the metropolitan circle. Some people may think this was not the only ground of their objection; but, however that may be, the Committee ruled against them and passed the Bill. This Committee, of which Lord Donoughmore was chairman, have been very energetic, for, besides dealing with these two schemes, they have disposed of an Omnibus Bill promoted by the South-Eastern Railway Company for constructing certain new works at Beckenham, and extending for two years the time previously allowed it for purchasing lands for improving the Charing-cross and Cannon-street Stations. The first proposal the Committee rejected, and, influenced mainly by the Metropolitan Board of Works, they limited the extension of time to one year only. In the two Chambers, before the adjournment until Monday next, several private Bills passed the second or third readings without opposition, but none of them require special notice.

A FAST YACHT.

THE *Mechanical Engineer* describes the Stiletto, a steam yacht built by the Herreshoff Company, and claims that this craft is the fastest afloat.

She is 95ft. long over all, by 11ft. beam, and 7ft. 9in. depth of hold. Her model is peculiar, and all that we can say about it is what the Herreshoff Company wish to make public; her bow lines are nearly straight, and she is sharp at both ends, with a round bottom; proportion of beam to length $8\frac{1}{4}$ ths. The engine is of the Herreshoff pattern with 12in. and 21in. cylinders by 12in. stroke, driving a 4ft. screw with 6ft. 6in. pitch. The engines can drive this 450 revolutions per minute, if required. The boilers are of the Herreshoff pattern, and to those unacquainted with them need a brief description. They are "pipe boilers" so called, and are made in sections like steam radiators. These sections are disposed over and around the furnace, and any one of them can be quickly detached, in case of need, for repairs. The water is fed in at the top and is converted into steam in its passage down, and emerges into a stand pipe or separator at one side of the sections; from this it goes to the engines. The advantages of this system are great lightness and efficiency. The whole boiler is heating surface, and being without a shell is capable of carrying very high pressure. The Stiletto works under 150 lb., that being sufficient to attain great speed, but we presume the boilers would stand 200 lb. just as well. Our readers at a distance may be interested in knowing that Mr. John B. Herreshoff, the principal of the Herreshoff Company, is totally blind, and has modelled all his vessels by the sense of touch alone; he has literally felt the lines out.

The fastest vessel that swims to-day, irrespective of size, is doubtless the steam yacht Stiletto. She recently ran from this city to Sing Sing on the Hudson, 30 miles by the chart, in 77 minutes from the start. This is an average of 23.7 miles per hour, but is not an expression of her highest velocity, for she was not put up to speed from the start, but ran some miles under easy steam. The Stiletto outran on this occasion the Mary Powell, hitherto the fastest boat on the river, beating her with ease. The Mary Powell was fully alive to the occasion, and did her best, but that was not enough. The Stiletto has achieved by this adventure a wide notoriety.

In closing, we may give the secret—if it may be so called—of the Stiletto's great speed. This results from her lightness, great engine power, and fine lines; but the first two qualities are paramount. She weighs only 28 tons and develops 450-H.P., having 16-H.P. per ton of displacement. This, as all engineers and constructors know, is simply tremendous in its possibilities. The Stirling Castle, the fastest merchant vessel afloat, has only 3 to 3½-H.P. per ton of displacement, while ocean steamers generally have only from 1 to 2-H.P. per ton of displacement.

THE SHEFFIELD INDUSTRIAL EXHIBITION.—The event of the week in the Sheffield district has been the visit of Prince Albert Victor of Wales, K.G., to open the Industrial Exhibition, which is promoted by the Company of Cutlers of Hallamshire. This Exhibition is the outcome of an offer of prizes by the master cutler, Mr. J. E. Bingham, for excellence in handicraft by Sheffield artisans. The project was cordially endorsed by the Cutlers' Company and the Chamber of Commerce, with the result that in a short time over £750 was subscribed for prizes, and a guarantee fund of £2000 to cover the expenses was also contributed. The Exhibition embraces every handicraft carried on within the boundaries of Hallamshire.

HENRI TRESCA.—It is with much regret that we record the death of M. Henri Tresca, an eminent French physicist and mechanical engineer. He was born at Dunkirk in 1814. He studied at the Polytechnic School, and on leaving it entered the corps of the Ponts et Chaussées, but soon afterwards quitted the service in order to devote himself to scientific study. In 1850 he was appointed principal inspector of the French Section of the Exhibition at London, and afterwards became sub-director of the Conservatoire des Arts et Métiers, and he there filled with great distinction the Chair of Industrial Mechanics. In 1872 he was elected a member of the French Academy. Of his numerous works may be mentioned his "Cours de Mécanique Appliquée" and his "Ecoulement des Liquides." Tresca was a very able physicist, and his name will ever be remembered in connection with his original research on the flow of solids. The Academy of Sciences, on hearing of his death from the President, M. Boulay, closed the sitting as a mark of grief. The Royal Society catalogue of scientific papers contains twenty-one references to important papers by Tresca. They include the determination of the coefficient of elasticity of aluminium; the test method of using the Prony brake in testing machines; several memoirs on the flow of solids, including ice and metals; on the application of the flow of solids in rolling and forging metals and the production of tubes; on the flexure of rails; on the properties of different bronzes; on prolonged torsion beyond the limit of elasticity; and on the mechanical equivalent of heat.

